

CALIBRATION OF POWDER CONSTITUTIVE MODEL USING DIGITAL IMAGE  
CORRELATION VALIDATED FOR HOLLOW HEMISPHERE OF LEAD ZIRCONATE  
TITANATE

Mateus Mota Morais<sup>a\*</sup>

Caiuã Caldeira de Melo<sup>b</sup>

Rodrigo Bresciani Canto<sup>b</sup>

Carlos Alberto Fortulan<sup>a</sup>

<sup>a</sup>Department of Mechanical Engineering, São Carlos School of Engineering, University of São Paulo, Trabalhador São-carlense, 400, São Carlos, 13566-590, Brazil

<sup>b</sup>Federal University of São Carlos (UFSCar), Department of Materials Engineering (DEMa), 13565-905 São Carlos, SP, Brazil

\*Corresponding author at: Department of Mechanical Engineering, São Carlos School of Engineering, University of São Paulo, Trabalhador São-carlense, 400, São Carlos, 13566-590, Brazil.

E-mail: mateus.morais@usp.br

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## ABSTRACT

This paper presents the application of digital image correlation to uniaxial and diametral compression tests to identify the Drucker-Prager/cap shear failure and the elastic parameters of a lead zirconate titanate powder (PZT). The calibrated parameters were validated through finite element simulation of the cold isostatic pressing of a hollow hemisphere used in piezoelectric transducers. The results showed that the dilatancy stresses were 50-65% lower than the fracture stresses, demonstrating that adopting the fracture stress may overestimate material cohesion. The simulation using the calibrated model achieved an excellent agreement with the measured geometry, with a difference of only 1.1% in the dimension and 2.0% in the average density. The results highlight how digital image correlation can obtain more accurate parameters with

fewer tests. The proposed methodology can be used to calibrate other powder materials, especially those in which dilatancy plays a key role.

**Keywords:** Drucker-Prager/cap; Powder compaction; Finite element; Parameter Identification; PZT; Dilatancy

## 1. INTRODUCTION

The finite element (FE) method has been widely used in academia and industry to model compacted powder parts and optimize their production and handling [1–4]. When effectively used, FE simulations can predict the dimensions, shape, density distribution, and possible failure regions of compacted parts [3,5,6]. These simulations improve designing molds for powder pressing and planning subsequent steps such as green machining [3,6,7], thus obtaining compacted parts with correct dimensions and uniform density, reducing trial-and-error iterations [8]. Especially for brittle materials, such as ceramics, simulations are fundamental to indicate regions more susceptible to cracks caused by non-homogeneous density distribution or excessive shear stresses [9–11].

For powder-like materials, there are two possible modeling strategies: the discrete approach (which considers each particle and its interactions individually) and the phenomenological or continuum approach (which considers the powder as a continuum) [9]. The continuum approach is usually implemented in FE simulations as it requires less computational effort and is a valid assumption when the size of the compacted part is orders of magnitude larger than the particles [9]. One of the most used continuum constitutive models for granular-like materials is the Drucker-Prager/cap (DPC) model [12,13] that has been applied with success to the simulation of several materials, such as soil [14], wood chips [15], pharmaceutical excipients [4,6,7,16,17], cosmetics [18], metals [19–22], cemented carbides [23] graphite [8], and other ceramics [5,24].

However, due to its complexity, the DPC model requires the calibration of several parameters to obtain accurate and reliable simulations [2,24,25]. Some alternatives have been proposed to reduce the number and complexity of the calibration tests. For instance, many authors used inverse analysis, adjusting the model parameters to minimize the difference between the simulation and a simple test (such as uniaxial compaction in a closed die). The minimization problem could be solved by classical optimization algorithms [23,24,26], genetic algorithms [27,28], neural network [11], and even a manual “trial-and-error” calibration [21].

However, there are some limitations to the optimization approach. First, it requires some previous knowledge about the limits of the parameters, which might be unavailable for new or uncharacterized materials. For instance, there is no information about the calibration of lead zirconate titanate (PZT) powder with the DPC model. Second, due to the large number of parameters of the DPC model, multivariable optimization can result in an unrealistic set of parameters that do not represent the material behavior in experimental conditions different from the calibration. For example, Zhou et al. [26] used inverse analysis and obtained a set of parameters that accurately represented the material behavior during the loading stage but failed to represent it during unloading.

A promising solution to minimize the calibration challenges is through digital image correlation (DIC), a non-contacting full-field measurement based on acquiring pictures of a sample throughout an experiment [29–32]. Considered very versatile, DIC has been widely adopted in diverse applications in the last twenty years, including identifying parameters of constitutive models [30–32]. One of the significant advantages of the DIC is to obtain a full-field displacement measurement instead of the average displacement, hence obtaining more information from a calibration test, which can reduce the number and complexity of required calibration experiments.

This paper presents the application of DIC to calibrate the DPC model for lead zirconate titanate (PZT) powder, highlighting the advantages of its use, especially identifying the material dilatancy. The calibrated model was validated by comparing the dimensions and density of a hollow PZT hemisphere (a piezoelectric component used in acoustic transducers) obtained by cold isostatic pressing with the FE simulation of the compaction.

## 2. CONSTITUTIVE MODEL AND CALIBRATION ALTERNATIVES

### 2.1. The Drucker-Prager/cap constitutive model.

The DPC model used herein is defined by three yield surfaces (Figure 1): a linear shear failure surface [12,13], an elliptical cap yield surface that governs material densification [12,33], and a smooth transition surface to facilitate numerical implementation [12]. If the stress state on the material reaches the cap yield surface, the material undergoes plastic compaction, increasing its density and expanding the cap yield surface (hardening) [9,12,16,34]. Conversely, if the stress state reaches the shear failure surface, the material undergoes plastic volumetric

expansion (also known as dilation or dilatancy) [13], contracting the cap yield surface (softening) [9,12,16].

The shear failure, elliptical cap, and transition yield surfaces are defined by Equations 1, 2, and 3, respectively, which relate the Mises equivalent stress,  $q$ , with the equivalent pressure stress,  $p$  (negative of the first invariant of the stress tensor) [12]. Several parameters are required to define the plasticity model: material cohesion,  $d$ , material internal friction angle,  $\beta$ , cap eccentricity parameter (ratio of the elliptical axis),  $R$ , and the transition surface radius parameter,  $\alpha$  (usually lower than 0.05). The dependent parameter,  $p_a$ , represents the abscissa value of the intersection of the cap and shear (or transition) surfaces (Equation 4). Furthermore, the model requires a hardening law of the cap surface determined by the relation of the hydrostatic compression yield stress,  $p_b$ , with the volumetric plastic strain,  $\varepsilon_{vol}^{pl}$ , which can be related to the material density by Equation 5, where  $\rho_0$  is the initial density (loose powder) and  $\rho$  is the current density. Finally, within the region limited by the yield surfaces, the material can be modeled with an isotropic linear elastic behavior (defined by Young's modulus,  $E$ , and Poisson's ratio,  $\nu$ ) [12,18,24,34,35]. The plastic flow potential of the DPC model is implemented in the software ABAQUS using a continuous and smooth plastic potential surface composed of two portions: an associated potential for the cap surface,  $G_{cap}$  (identical to the cap yield surface,  $F_{cap}$ , in Equation 2) and a nonassociated potential in the shear failure and transition surfaces,  $G_{shear}$  (Equation 6) [12,16].

$$F_{shear} = q - p \tan \beta - d = 0 \quad (\text{Eq. 1})$$

$$F_{cap} = \sqrt{(p - p_a)^2 + \left[ \frac{Rq}{1 + \alpha - \alpha / \cos \beta} \right]^2} - R(d + p_a \tan \beta) = 0 \quad (\text{Eq. 2})$$

$$F_{tran} = \sqrt{(p - p_a)^2 + [q - (1 - \alpha / \cos \beta)(d + p_a \tan \beta)]^2} - \alpha(d + p_a \tan \beta) = 0 \quad (\text{Eq. 3})$$

$$p_a = \frac{p_b - Rd}{1 + Rt \tan \beta} \quad (\text{Eq. 4})$$

$$-\varepsilon_{vol}^{pl} = \ln(\rho / \rho_0) \quad (\text{Eq. 5})$$

$$G_{shear} = \sqrt{[(p_a - p) \tan \beta]^2 + \left[ \frac{q}{(1 + \alpha - \alpha / \cos \beta)} \right]^2} \quad (\text{Eq. 6})$$

## 2.2. Alternatives for calibration of the Drucker-Prager/cap model

One of the great challenges of using the DPC constitutive model is the number and complexity of the experiments required to calibrate its parameters. The calibration of the shear failure surface parameters ( $d$  and  $\beta$ ) requires at least two tests with distinct stress states. Figure 1 displays four alternatives: uniaxial tension (1), pure shear (2), diametral compression (also known as Brazilian disk test or diametral compression) (3), and uniaxial compression (4) [8,9,12,16,24,35]. Due to the fragile nature of compacted powder materials, diametral and uniaxial compression tests are the most common choices [8,36]. Incorrect calibration of the shear failure parameters can lead to unrealistic simulation results, especially during the unloading and ejection of powder compacts [25,29]. Therefore, it is fundamental to correctly identify the dilatancy of the granular material [29], which is characterized by a permanent increase in the volume caused by shear stress, leading to micro-cracks and material unpacking. [12,13,29]. Identifying plastic dilatancy initiation is still a topic that needs more investigation and discussion [8]. When calibrating the DPC model, most researchers adopted the peak stress (fracture stress) to define the onset of plastic deformation [16,18,21,22,37,38], while others adopted the deviation of linearity [8]. However, it is known that the fracture stress is higher than the actual dilatancy stress [29]. Proper identification of dilatancy would require measuring the volumetric strain, which is difficult using conventional extensometers.

Moreover, the calibration of the cap yield surface parameters ( $R$ ,  $\alpha$ , and  $p_b(\varepsilon_{vol}^{pl})$ ) also commonly requires two tests at different stress states [24], though some authors obtained good results adopting only one test based on reasonable assumptions concerning the plastic flow [16]. The first test is an incremental isostatic pressing test used to obtain the hardening law  $p_b(\varepsilon_{vol}^{pl})$  (Test 6 in Figure 1) [2,12,24,39]. A second test with a stress state that lies in the cap surface is necessary to obtain the cap eccentricity  $R$  (Test 5 in Figure 1). This stress state can be obtained through complex triaxial tests [2,40] or a closed die compaction test in an instrumented die (an experimental setup that measures the upper and lower punch loads and the radial pressure in the die wall) [9,16,17,19,22,36,41,42]. The instrumented die has the advantage that it can also be used to determine the friction coefficient,  $\mu$ , between the powder and the die [35,41,42], but it is not available in many laboratories. Finally, it is unnecessary to have a calibration experiment for the cap transition parameter,  $\alpha$ , because it is an artificial parameter used only to avoid numerical instabilities. Thus, it can be set to the lowest possible value that does not cause numerical instabilities without significantly impacting the simulation results [43].

The elastic parameters should also be calibrated. The Young's modulus,  $E$ , can be easily obtained with conventional uniaxial compressive tests. On the other hand, obtaining material

Poisson's ratio,  $\nu$ , is more complicated due to the greater difficulty in measuring transversal strain. However, it is crucial to obtain its value because Poisson's ratio strongly influences residual stresses during and after sample ejection [44]. Most authors obtained the elastic parameters during the unloading stages of instrumented closed die compaction tests [16,18,34,35].

Furthermore, it has been shown that all previous parameters can be considered dependent on the compacted density [16]. Therefore, if an accurate prediction of density and stress distribution in the compacted part is required throughout all the simulation stages, the calibration tests previously presented should be repeated for several densities, further increasing the number of required experiments [16].

A possible solution to minimize the number of experiments and maximize the information obtained through them is using DIC. Because it can obtain the full-field measure of displacement, DIC can be used to calculate both the vertical and horizontal strains, thus the volumetric strain, allowing to precisely identify Poisson's ratio and the onset of the dilatancy. For instance, Montilha et al. [29] successfully determined the dilatancy stress of a compacted alumina sample by applying DIC to a cyclic uniaxial compression test. Additionally, DIC can obtain local strain, subtracting the artifacts caused by the testing machine compliance, reducing calibration errors. Further, DIC can identify non-homogeneous strain fields and defects in the sample caused by undesired bending [45] or cracks [46], which might be unnoticed otherwise.

### 3. MATERIALS AND METHODS

#### 3.1. Calibration of the Drucker-Prager shear surface parameters.

The material used in this study was PZT powder (SP-4, Navy I type) with binder as supplied by the Brazilian Navy Research Institute (IPqM). The median particle size was  $d_{50} = 3.1 \mu\text{m}$ , the weight loss on ignition was 0.9 % (350 °C for 2 h), and the helium pycnometer density was 8.00 g/cm<sup>3</sup>. The bulk density of the loose powder was 2.75 g/cm<sup>3</sup>, which was measured volumetrically by pouring and leveling the powder in a graduated cylinder.

To calibrate the shear surface, DIC was applied to identify the dilatancy on uniaxial compression and diametral compression tests, in a similar manner to what the authors reported in previous studies [29,45]. Three specimens were used for each test. For the uniaxial compression tests, two-dimensional image correlation requires planar faces [32]. Thus, parallelepiped specimens were produced using a hardened steel die with a square hole (18.5 x

18.5 mm) cut by electrical discharge machining and steel punches that were ground and polished to improve the flatness and orthogonality of the faces. The uniaxial compression specimens were pre-compacted uniaxially at 60 MPa in this parallelepiped cavity steel die. For the diametral compression tests, three cylindrical disk specimens were pre-compacted uniaxially at 45 MPa in a cylindrical cavity steel die (inner diameter of 41 mm). To reduce the compaction gradients during pre-compaction, the walls of the molds were lubricated with oleic acid and foam pads were used to support the base of the molds during compaction to mimic the effect of a double-action pressing. After uniaxial pre-compaction, all specimens were isostatically pressed at 200 MPa inside vacuum-sealed thin latex elastomers. A random black-and-white speckle pattern was sprayed onto the specimen surfaces to increase the image contrast and improve DIC resolution [24,45,47,48]. To ensure an adequate contrast, the gray-level histograms of the images were analyzed, where the randomization and dynamic range were verified. Figure 2 displays the molds and the pre-compacted specimens. Tables 1 and 2 display the dimensions and mass of the uniaxial and diametral compression test specimens, respectively.

Table 1 – Dimensions and mass of uniaxial compression test samples.

Specimen	Edge mm	Height mm	Mass g
U1	17.95	33.84	57.88
U2	17.97	36.14	61.96
U3	17.94	40.25	68.93

Table 2 – Dimensions and mass of diametral compression test samples.

Specimen	Diameter mm	Thickness mm	Mass g
D1	39.98	6.03	39.77
D2	40.02	5.70	37.76
D3	40.07	6.00	39.75

The experimental setup for uniaxial compression text is displayed in Figure 3. A similar setup was used for the diametral compression test, changing only the location of the cameras. The tests were performed at an electromechanical universal testing system (MTS Exceed Model E44 with 30kN load cell) (Fig. 3, label 1). The upper and lower parts of the specimens were fixed and aligned on the loading plates (Fig. 3, label 2) using epoxy resin. Figure 4 shows the speckled specimens positioned for the tests. The photos were acquired simultaneously using two cameras (Canon EOS 5DS - 8688 x 5792 pixels, 16bits with EF 180mm f/3.5L Macro USM lenses - Fig. 3, label 3), which were used to check if the strains were similar in two faces of the

specimens, validating the assumption of negligible bending load [45]. White LED was used for specimen illumination (Fig. 3, label 4). The cameras and the illumination LEDs were fixed in rigid supports (Fig. 3, label 5), which were leveled and aligned. The cameras were positioned with the aid of the open-source control software digiCamControl on a laptop (Fig. 3, label 6) connected to the cameras, where the images were verified and recorded. The distance between the cameras and the specimen was about 0.6 m, measured from the camera sensor. Finally, the cameras were triggered by the testing machine controller (Fig. 3, label 7) whenever an established load variation (positive or negative) was exceeded, resulting in an acquisition rate between 0.02 and 0.2 frames per second (fps).

In the uniaxial compression test, the cameras were positioned perpendicular to each other, aligned with two lateral faces of the parallelepiped specimens (Figure 5A). In the diametral compression test, the cameras were positioned facing each other, aligned with the flat faces of the disk specimens (Figure 5B). The cameras were focused on the central area of the specimens, where uniform strain fields were expected. For both tests, the physical size of a pixel was about 9  $\mu\text{m}$  (which was calibrated comparing the number of pixels in the image with the dimensions of the specimens).

The first specimen of each type of test was loaded continuously until fracture to obtain the overall behavior of the loading curve and the fracture stress. The second and third tests consisted of four incremental loading-unloading cycles followed by a final loading until the specimen fractured. Cycled tests allow tracking the plastic strain evolution during unloading, which can be used to identify the dilatancy [29]. A smaller load trigger was used in the second and third specimens of both tests to obtain more detailed loading and unloading curves. Table 3 contains the parameters used in the tests.

Table 3 – Parameters of the uniaxial and diametral compression tests.

Test	Crosshead speed [mm/min]	Load variation trigger for each specimen			Preload [N]	Maximum load in each cycle			
		1°	2°	3°		1°	2°	3°	4°
Uniaxial	0.1	70	40	45	20	300	600	1000	1500
Diametral	0.05	20	5	5	10	30	60	90	120

The DIC was processed using Matlab<sup>TM</sup> with the Correlli 3.0 framework, developed at LMT ENS Paris-Saclay [49]. Also called global DIC, this algorithm aims to obtain the displacement field  $\mathbf{u}$ , globally minimizing the difference of the gray levels between a reference image  $\mathbf{f}$  and a deformed image  $\mathbf{g}$  corrected by the measured displacement. The cost function (Equation 7) is used to describe the gray-level conservation law, where  $N_i$  is the number of

pixels in the region of interest and  $\mathbf{x}_i$  is the coordinate vector of the  $i$ -th pixel. To guarantee good conditioning of the minimization, the displacement of the  $i$ -th pixel,  $\mathbf{u}_i$ , is described by a set of degrees of freedom via finite element (FE) shape functions (Equation 8), where  $\mathbf{a}_j$  are the sought nodal displacements,  $\Phi_j(\mathbf{x}_i)$  are the FE shape functions, and  $N_j$  is the number of nodes.

$$\eta^2 = \sum_i^{N_i} [f(\mathbf{x}_i) - g(\mathbf{x}_i + \mathbf{u}_i)]^2 \quad (\text{Eq. 7})$$

$$\mathbf{u}_i = \mathbf{u}(\mathbf{x}_i) = \sum_j^{N_j} \Phi_j(\mathbf{x}_i) \mathbf{a}_j \quad (\text{Eq. 8})$$

Before each test, an initial picture was acquired for each camera to calibrate the reference image  $f$ . The displacement field was calculated using three-node triangular FE shape functions (T3-DIC) [29,46,47]. A mesh of triangular elements (about 100 elements for uniaxial compression tests and 1200 elements for diametral compression tests) was generated in the region of interest in the center of the pictures. Then, the strains were calculated for each element using the nodal displacements obtained by DIC and the triangular FE shape functions. Because no discontinuities in the strain field were detected, the vertical and horizontal strains at each photo were calculated considering the average strain field of the mesh elements in the analyzed area. Plain stress was assumed in the diametral compression test because the disk diameter was more than five times larger than its thickness [36]. Table 4 presents the expressions used for calculating the volumetric strain ( $\varepsilon_{vol}$ ), the nominal stresses ( $\sigma_{xx}$ ,  $\sigma_{yy}$  and  $\sigma_{zz}$ ), the equivalent pressure ( $p$ ) and the equivalent von Mises stress ( $q$ ), where  $F$  is the applied force measured by the testing machine,  $A$  is the cross-sectional area of the parallelepiped specimens,  $D$  and  $t$  are the diameter and the thickness of the disk specimens, respectively. These values were calculated for each picture, considering the average strains. In the uniaxial compression test, the horizontal strains ( $\varepsilon_{xx}$  and  $\varepsilon_{zz}$ ) were assumed equal, while in the diametral compression test, the strain in the direction of the disk thickness ( $\varepsilon_{zz}$ ) was assumed negligible. The deduction of the diametral compression stress equations [50,51] and the  $p/q$  ratio [16] can be found in literature.

Table 4 – Expressions for stresses and volumetric strain of the uniaxial and diametral compression tests.

Test	$\varepsilon_{vol}$	$\sigma_{xx}$	$\sigma_{yy}$	$\sigma_{zz}$	$p$	$q$
Uniaxial	$\varepsilon_{yy} + 2\varepsilon_{xx}$	0	$\frac{F}{A}$	0	$\frac{-\sigma_{yy}}{3}$	$-\sigma_{yy}$
Diametral	$\varepsilon_{yy} + \varepsilon_{xx}$	$\frac{-2F}{\pi D t}$	$-3\sigma_{xx}$	0	$\frac{2\sigma_{xx}}{3}$	$\sqrt{13}\sigma_{xx}$

The elastic constants were obtained considering only the first cycle of the uniaxial compression tests U2 and U3. As discussed in the results section, the second, third, and fourth cycles already presented some plastic deformation, modifying the elastic constants. The Young's modulus was obtained from the slope of the linear fit of the vertical stress ( $\sigma_{yy}$ ) by vertical strain ( $\varepsilon_{yy}$ ) curves. Similarly, the Poisson's ratio was obtained from the slope of the linear fit of the horizontal strain ( $\varepsilon_{xx}$ ) by vertical strain ( $\varepsilon_{yy}$ ) curves.

The critical dilatancy stress was defined as the Mises stress occurring at the minimum volumetric strain, as any additional load caused volumetric dilation. The dilatancy stress and the fracture stress of all tests were plotted in the  $q$  vs.  $p$  graph. A linear fit of the failure surface was calculated for the dilatancy and the fracture stresses. The shear failure constants ( $d$  and  $\beta$ ) were obtained as the linear and angular coefficients of the regression, respectively.

### 3.2. Calibration of the hardening and the cap surface

An incremental hydrostatic compression test was used to calibrate the cap hardening curve. The PZT powder was pressed uniaxially at 10 MPa in a cylindrical cavity steel die (38 mm diameter – sample H1). Immediately after uniaxial pressing, the mass of the disk was measured. The disk was packed inside a vacuum-sealed elastomer and was isostatically pressed at the following pressures: 20, 40, 60, 80, 120, 160, and 200 MPa. After each pressing stage, the specimen was depressurized, unpacked, and its dimensions were measured before the next pressing stage. The density and the equivalent volumetric plastic strain were calculated for each pressure stage. Two additional replicas (H2 and H3) of the experiment were performed, with slight modifications: die cavity of 41 mm diameter, initial uniaxial pressure of 15 MPa and final isostatic pressure of 190 MPa.

The hardening law was adjusted by an exponential fit [21] with a linear term (Equation 9). The inclusion of the linear term avoided numerical problems in the simulation that arise when the hydrostatic compression yield stress ( $p_b$ ) does not satisfy  $p_b > Rd$  (equivalent to  $p_a > 0$ ) at the beginning of the simulation. The linear term  $c_3$  was fixed in 0.8 to avoid these numerical problems and satisfy  $p_b > Rd$  for both dilatancy and fracture simulations.

$$p_b = c_1 e^{\left(c_2(-\varepsilon_{vol}^{pl})\right)} + c_3 \quad (\text{Eq. 9})$$

The transition parameter,  $\alpha$ , was set the lowest recommend value of 0.01, as it has a negligible effect on the simulation results [12,43]. The cap eccentricity,  $R$ , and the friction coefficient,  $\mu$ , were adopted from previous research with alumina powder [24], another ceramic material. Despite being another material, the cap eccentricity and friction coefficient have less influence when the expected stress field is predominantly hydrostatic, as in the proposed validation study. Furthermore, when comparing the available data for other powder materials, the friction coefficient usually varies between 0.1-0.2 for metallic powders [11,20–22,42,52], 0.2-0.3 for ceramic powders (Alumina [3,24,53], Silicon Carbide [54], and zirconia [55]), and up to 0.45-0.48 for pharmaceutical powders (lactose [4] and microcrystalline cellulose [41]). PZT particles, like other ceramic materials, are mechanically hard and stiff (large Young's modulus) and are expected to behave similarly in friction. Usually, larger friction coefficients enhance shear stresses, increasing shear failure (dilatancy and fracture). Therefore, based on previous research with alumina, the upper limit found in ceramics ( $\mu = 0.3$ ) [24] was adopted as the friction coefficient between the PZT powder and the unlubricated metallic core.

### 3.3. Case study: isostatic pressing of a hollow hemisphere

A case study was carried out to validate the calibrated parameters of the constitutive model. The proposed geometry was a hollow hemisphere of PZT used as the piezoelectric component of an acoustic transducer. This geometry has a low thickness-to-diameter ratio, making it more susceptible to warping and cracking if the density field is not uniform [3]. Therefore, isostatic pressing is the recommended compaction strategy to obtain parts with high and uniform density [56]. The isostatic mold comprised four pieces: a stainless-steel core, a silicone rubber bag with its cover (to seal the mold), both made of RenCast 4644-1/Huntsman - hardness 41 Shore A, and a 3D-printed ABS polymer jacket, which served to hold the mold components together. The mold was designed with a hemispherical cavity with a cylindrical region with 5 mm height in the base to separate the hemispherical region from the mold base, where density gradients usually occur. Figures 6A and 6B display a digital model of the mold with a sectional cut to reveal its interior and dimensions. Figure 6C displays a photo of the components of the mold.

The proposed geometry was well-suited to the validation study for some reasons. First, the geometry and loadings were distinct from what was used during calibration. Second, the stress conditions in this geometry were mainly hydrostatic during compaction, minimizing the effect of cap eccentricity,  $R$ , which had not been calibrated for PZT powder. Finally, although the compaction was isostatic, the friction of the powder with the metallic core created shear stresses during unloading, validating the shear surface.

The hemisphere was produced by isostatic pressing at 200 MPa using 27.0 g of PZT powder. The mold was manually fed using a vibratory shaker table, thus increasing the density of the powder inside the mold compared to the loose bulk density. As a result, the initial plastic volumetric strain ( $-\varepsilon_{vol}^{pl}|_0$ ) was higher than zero. This parameter of the constitutive model represents the initial hardening of the model and controls the cap surface position at the beginning of the simulation. The initial density of the powder (3.70 g/cm<sup>3</sup>) was calculated volumetrically, considering the mold cavity volume (7.3 cm<sup>3</sup>), which resulted in an initial plastic volumetric strain of -0.295 calculated according to Equation 5. Two hemispheres had their external profile measured on a Nikon V-20A profile projector using the 10x magnifying lens. The inner diameters of twenty hemispheres were measured using a three-dimensional measuring machine. The densities of seven fragments of the hemispheres were measured by immersion, based on Archimedes' principle. The method was adapted to use isopropyl alcohol instead of water to avoid destroying the compacted samples [57] because the binder supplied with the powder was water-soluble (probably polyvinyl alcohol).

### 3.4. Finite element model validation

The simulations of the isostatic pressing of the hollow hemisphere were performed in ABAQUS standard solver with unsymmetric matrix storage to account for the possible unsymmetric material stiffness caused by the nonassociated portion of the plastic flow potential of the DPC model [12]. The PZT powder was modeled using the calibrated DPC model. Three simulations were performed: 1) model with the dilatancy parameters **and** initial hardening; 2) model with the dilatancy parameters **without** initial hardening; 3) model with **fracture** parameters and initial hardening. The purposes of the second and third simulations were to evaluate the effect of the initial hardening and the shear parameters, respectively.

The metallic core was modeled as a rigid surface because the ceramic powder would undergo much larger deformation. It was assumed that the silicone rubber bag would not

significantly influence the simulation because the applied loads are mainly isostatic. Due to geometry and loads symmetry, an axisymmetric model was built. It was assumed that the mold would be overfilled with powder, with a 1 mm height column of powder in the feed channel of the mold. The powder mesh was built with 870 CAX4R elements. Contact was defined using the contact pair algorithm with finite-sliding formulation, surface-to-surface discretization, hard contact normal behavior, and Coulomb friction of 0.3. The metallic core was defined as the master surface.

The degrees of freedom of the metallic core were fully constrained. A radial symmetry boundary condition was imposed on the model. The simulation was performed in three subsequent steps: Initial (application of boundary conditions and contact stabilization); Loading (200 MPa pressure gradually applied to the external surface of the powder); and Unloading (99.7% of pressure gradually removed). Total unloading was not possible due to numerical convergence difficulties related to the non-associated plastic flow potential of the shear failure surface of the DPC model. The FE model is displayed in Figure 7. The ABAQUS input file of simulation 1 (dilatancy parameters with initial hardening) is presented in the supplementary material S1. The density field was calculated with the volume of each element (variable field named EVOL in ABAQUS) according to Equation 10, where  $EVOL_0$  and EVOL are the element volume in the initial and current increment of the simulation. The calculation was performed through a macro script written in Python language presented in the supplementary material S2.

$$\rho = \rho_0 (EVOL_0 / EVOL) \quad (\text{Eq. 1})$$

## 4. RESULTS AND DISCUSSIONS

### 4.1. Calibration results

The strain fields obtained by DIC in the uniaxial and diametral tests were uniform, evidencing that the specimens had no discontinuities caused by cracks or undesired bending, validating the initial assumption of uniform strain in the calculations. Furthermore, discontinuities caused by crack initiation were observed only near the end of the tests when the specimens fractured (as expected).

The uniaxial compression test stress-strain curve is presented in Figure 8. The incremental load cycles evidenced the progressive plastic deformation. As previously discussed, the elastic parameters were obtained considering only the first cycle. It can be

observed in Figure 8 that only the first cycle presented negligible plastic deformation, fully recovering the deformation. The calibrated elastic parameters are presented in Table 5.

The presence of significant plastic deformation in final cycles was an indicator that dilatancy had already taken place. To better visualize the effect of dilatancy, a graph of the deviatoric strain by volumetric strain in the uniaxial test is presented in Figure 9. During the initial stages of the test, the volumetric strain decreased (specimen volume reduced). However, as compaction continued, the volumetric strain reverted its decreasing tendency and started to increase. The volumetric strain significantly increased during the final cycle, exceeding the initial volume reduction and becoming positive (specimen volume increased). The dilatancy point was defined at the minimum volumetric strain (when the material started to expand due to the increasing load). The dilatancy stress was obtained from the equivalent Mises stress at this point.

Table 5 – PZT elastic parameters obtained in the uniaxial compression test and the respective coefficients of determination of the linear regressions.

Specimen	Camera	$E$ GPa	$R^2$	$v$	$R^2$
U2	1	1.70	0.96	0.180	0.96
U2	2	1.55	0.95	0.173	0.80
U3	1	1.81	0.94	0.195	0.75
U3	2	1.88	0.95	0.164	0.69
<b>Average</b>		<b>1.8</b>		<b>0.18</b>	
<b>Std deviation</b>		<b>0.1</b>		<b>0.01</b>	

Concerning the diametral compression test, a similar analysis was made. The vertical stress-strain curve of a diametral test is presented in Figure 10. The dilatancy point was identified in the deviatoric strain by the volumetric strain graph in Figure 11. In contrast to the uniaxial results, performing a cycled test of the diametral compression added no relevant information and made the analysis more difficult due to the larger number of points. For future studies, monotonical loading is recommended instead of cycled loading-unloading for the diametral test. Furthermore, diametral tests also presented more noise and larger deviation, highlighting the importance of using more than one sample.

The dilatancy stress and the fracture stresses for the three uniaxial compression tests are summarized in Table 6. The Drucker-Prager shear surfaces were obtained through a linear fit of the stresses of uniaxial and diametral compression tests. Figure 12 depicts the results in the  $p$ - $q$  plane considering both dilatancy and fracture.

Table 6– Dilatancy and fracture stresses at the uniaxial and diametral compression tests.

Specimen	Dilatancy	Fracture
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	MPa	MPa
U1	2.431	6.104
U2	2.401	6.765
U3	2.849	6.729
D1	0.918	2.029
D2	1.200	2.351
D3	0.806	1.630

It can be observed that the dilatancy stress is much smaller than the fracture stress. Thus, when the material is subjected to shear, unpacking and crack initiation occur before fracturing [29]. Consequently, using the dilatancy stress instead of fracture stress is more indicated when modeling compacted powder shear failure.

The hydrostatic compression test results and the exponential fit obtained for the cap hardening are presented in Figure 13. The constants of the exponential fit (Equation 9) were calculated as  $c_1 = 6.31 \cdot 10^{-4}$  MPa,  $c_2 = 21.2$ , and  $c_3 = 0.8$  MPa, with  $R^2$  of 97.1%. Table 7 summarizes all parameters of the Drucker-Prager/cap model used in the simulations. The parameter  $K$  is the yield stress ratio in triaxial tension to triaxial compression, and it is usually assumed as one [12]. Although no data about PZT Drucker-Prager/cap parameters were found in literature, the parameters obtained in this study were in a similar range to other powder materials (Table 8), evidencing a reasonable behavior.

Table 7 –Drucker-Prager/cap parameters for PZT in the simulations.

Simulation	$E$ GPa	$\nu$ -	$d$ MPa	$B$ Degrees	$R$ -	$-\varepsilon_{vol}^{pl} _0$ -	$\alpha$ -	$K$ -
1 – dilatancy with initial hardening	1.758	0.178	0.537	67.21	0.75	0.295	0.01	1
2 – dilatancy without initial hardening	1.758	0.178	0.537	67.21	0.75	<b>0</b>	0.01	1
3 – fracture with initial hardening	1.758	0.178	<b>1.06</b>	<b>68.31</b>	0.75	0.295	0.01	1

Table 8 –Drucker-Prager/cap parameters and friction coefficient between powder materials and die walls obtained from literature (for comparison).

Material	$E$ GPa	$\nu$ -	$d$ MPa	$\beta$ Degrees	$R$ -	$\alpha$ -	$\mu$ -	Reference
Microcrystalline cellulose (60% density)	1.0	0.2	1.0	72	0.6	0	0.26	[17]
Graphite	-	-	1.0	60	0.7	0	-	[8]
LiNiMnCoO <sub>2</sub> (NMC)	-	-	4.0	65	0.8	0	-	[8]
Alumina	2.7	0.13	3.25	55	0.75	0	0.3	[24]

## 4.2. Validation results

Figure 14 displays two compacted hemispheres. The pressed parts had an average external diameter of  $32.4 \pm 0.1$  mm and an inner diameter of  $26.48 \pm 0.01$  mm. Supplementary material S3 includes the measured profiles data. In contrast, in the simulation with dilatancy and initial hardening (Simulation 1), the part had an external diameter of 32.53 mm and an inner diameter of 26.78 mm, 0.4% and 1.1% larger than the measured dimensions, respectively. The dimensions were evaluated 5 mm above the base of the part (the junction between the hemispherical and cylindrical regions). The average density of the pressed parts was  $5.09 \pm 0.02$  g/cm<sup>3</sup> (63.5% of the theoretical maximum). In contrast, the simulated part density varied from 5.05 to 4.92 in the hemispherical region (the average was 4.99 g/cm<sup>3</sup>, 2.0% smaller than measured). The comparison of the measured and simulated cross-section profiles is depicted in Figure 15Figure 15, which also presents the simulated density field.

The simulated profile presented high similarity with the pressed part, visually and numerically. Both the measured and simulated inner diameters were slightly larger than the mold core diameter (26.36 mm). The gap between the metallic core and the part is displayed in Figure 16A. The elastic springback of the part caused this gap during unloading. The simulation predicted the springback accurately, pointing out that the calibration of the elastic parameters was validated. Even though the model partially included powder data from other ceramic material (alumina) ( $R$  and  $\mu$ ), the FE simulation accurately modeled the compaction process, validating the calibration procedures.

The differences in the results should also be highlighted. The measured part presented a thickness reduction from bottom to top. It can be observed in Figure 15 that the measured profile in the bottom was similar to the simulation with initial hardening (Simulation 1). On the other hand, the simulation without hardening (Simulation 2) better represented the compaction behavior in the upper region of the part. As a result of the mold filling process, the powder in the bottom was more tapped than the powder at the top. Consequently, the powder at the bottom had a larger initial density and suffered less volumetric compaction than the upper region, resulting in a thicker wall.

Furthermore, a defect known as “elephant foot” [58] was observed in the base of the part (Figure 16). This defect was caused by the friction of the powder with the mold base (which was intensified by the limited deformation of the rubber in the edge of the metallic core, as observed in the projected profile in Figure 16B). Consequently, the bottom of the part presented a less dense region with an intense density gradient, as observed in Figure 16A. Furthermore,

the similar shape of the bottom indicates that the assumption of  $\mu = 0.3$  is close to the actual friction coefficient of PZT powder.

Finally, Figure 17 compares the simulations with the fracture and dilatancy calibrated parameters. As can be observed, adopting the fracture data led to underestimating the regions where there was plastic volumetric expansion and damage initiation. Conversely, adopting the dilatancy data calibrated using DIC allowed more accurate identification of regions susceptible to failures. In the proposed geometry, the bottom edge and some external surface areas were the regions with dilatancy, smaller density, and more intense density gradient than the rest of the part. These regions may cause significant defects because density gradients greater than 1%/mm can introduce cracks or considerable warping during sintering [3]. For this geometry, green machining should be used to correct the non-uniform thickness of the part and to remove the damaged regions before sintering, improving the mechanical properties of the final part [3]. Therefore, the accurate identification of the critical regions is fundamental for planning the amount of material that should be removed during the green machine step. The proposed methodology for calibration of the DPC model using DIC allowed better identification of the onset of dilatancy, resulting in a more accurate simulation than what could be achieved using other calibration strategies.

## 5. CONCLUSIONS

The finite element (FE) simulation with the adjusted Drucker-Prager/cap (DPC) parameters allowed a deeper understanding of the pressing process, predicting the shape, dimensions, and density field of the pressed part. The design of molds can be notably aided by numerical simulation, as it reduces trial-and-error, saving time and resources. The use of digital image correlation (DIC) allowed obtaining more information from the uniaxial and diametral compression tests, including the Poisson's ratio. Remarkably, because DIC is a full-field strain measurement, the proposed methodology was able to identify the onset of dilatancy, which occurs in lower stress than fracture, and may not be obtained with traditional calibration techniques. Therefore, the shear failure surface parameters of the DPC model were calibrated with improved accuracy. With the additional information obtained by DIC, only three types of simple tests (uniaxial compression, diametral compression, and volumetric compression) resulted in a near-complete calibration of the DPC model, without complex triaxial tests or instrumented dies. Despite no calibration of the cap eccentricity,  $R$ , and friction coefficient,  $\mu$ , the numerical validation showed significant agreement with the experimental measures with a

difference of only 1.2% in the internal diameter and 2.2% in the density, highlighting the success of DIC to obtain accurate results, especially for near isostatic pressing. The method proved to be an efficient tool for predicting critical regions susceptible to deformation and cracking during the sintering shrinkage, allowing better planning of the green machining step. The proposed methodology can be used to calibrate other powder materials in which dilatancy plays a key role. In future developments, the constitutive model of the powder calibrated in this work has a potential application in the design of molds for other PZT components with dimensions or geometries different from those used in this paper.

Some concerns about the limitations of the present study and possible alternatives should be noted. In this paper, the DPC model was considered density-independent. If a density-dependent model had been chosen, the parameters should have been calibrated at several density values [16,35,38,59]. Nevertheless, the DIC methodology could be easily extended to calibrate a series of uniaxial and diametral compression tests with specimens with incremental densities. The full-field measure could also be used to calibrate non-linear elastic parameters, implementing more complex DPC models such as proposed by Han et al. [16]. Furthermore, despite the excellent validation results, the calibrated parameters did not include the  $R$  and  $\mu$ . There are two possible alternatives that do not require instrumented die nor triaxial tests. First, ZHU et al. [8] used DIC to perform closed die compaction and measure the radial and axial deformation of the die, thus obtaining similar results to an instrumented die. This alternative takes advantage of the DIC equipment used to calibrate the shear failure surface, allowing full calibration of the DPC model. The second alternative is to combine the DIC calibration with inverse analysis optimization to obtain the missing parameters [24]. The DIC calibration has the advantage of reducing the number of parameters required for the inverse analysis, improving its accuracy, and reducing its computational cost.

## CRediT AUTHOR STATEMENT

Mateus Morais: Experimental investigation, Numerical analysis, Writing – original draft preparation. Caiuã Melo: Experimental investigation, Software – supporting algorithms, Writing – reviewing and editing. Rodrigo Canto: Conceptualization, Writing – reviewing and editing, Funding acquisition. Carlos Fortulan: Writing – reviewing and editing, Supervision, Funding acquisition.

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## DECLARATION OF COMPETING INTEREST

The authors declare that they have no known competing financial interests or personal relationships that could have appeared to influence the work reported in this paper.

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## FIGURES

Figure 1 – Drucker-Prager/cap model and possible tests to calibrate the yield curves. Adapted from [9,12,16].

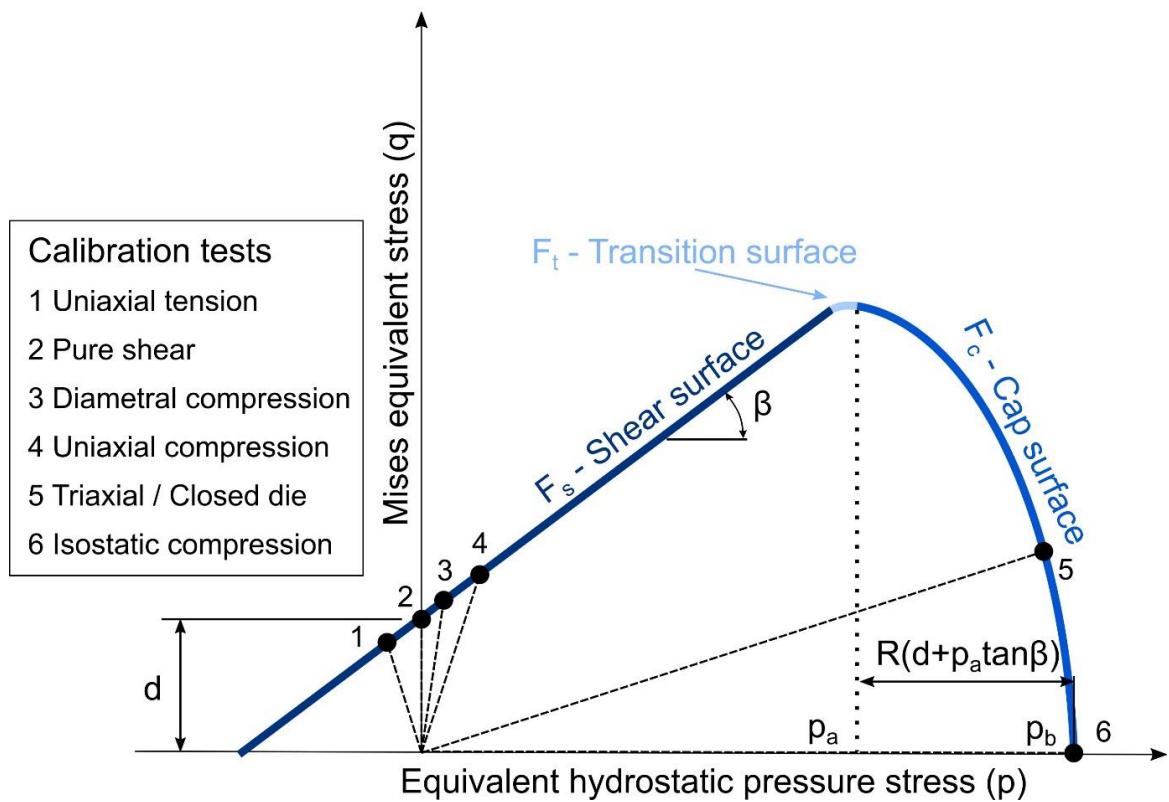


Figure 2 – Photos of molds and pre-compacted specimens. A) Samples for uniaxial compression. B) Samples for diametral compression

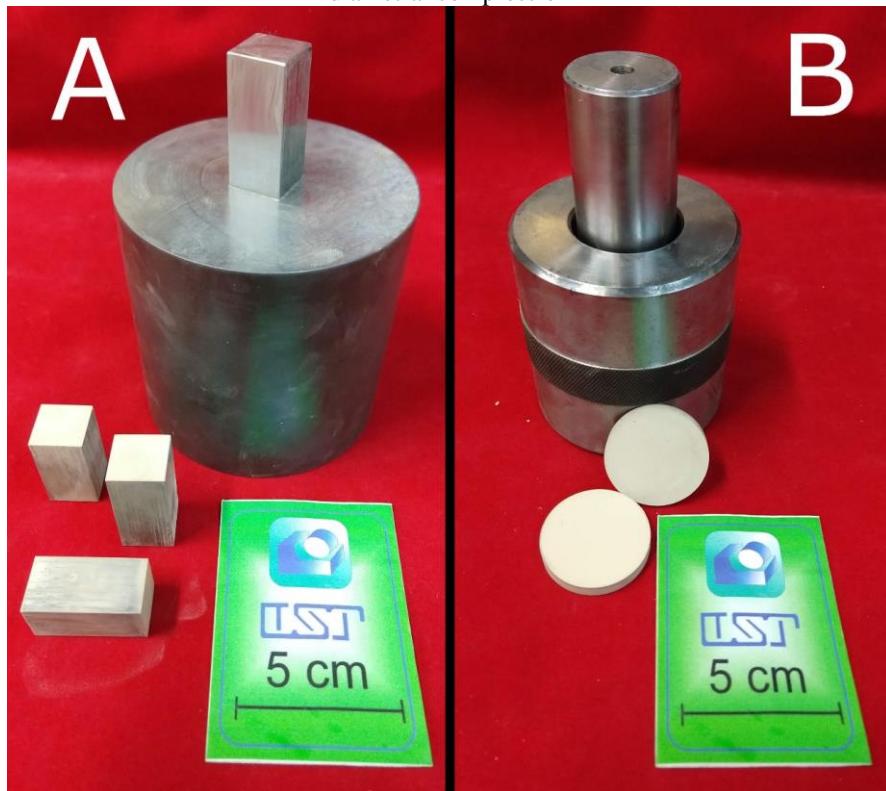


Figure 3 – Photos of the experimental setup for the uniaxial compression test. In A, frontal view. In B, lateral view. Legend: 1 – Universal testing machine; 2 – Loading plates; 3 – Digital cameras with Macro lens; 4 – LED lights; 5 – Leveled rigid supports; 6 – Laptop with software for controlling the cameras; 7 – Computer with the test machine controller.

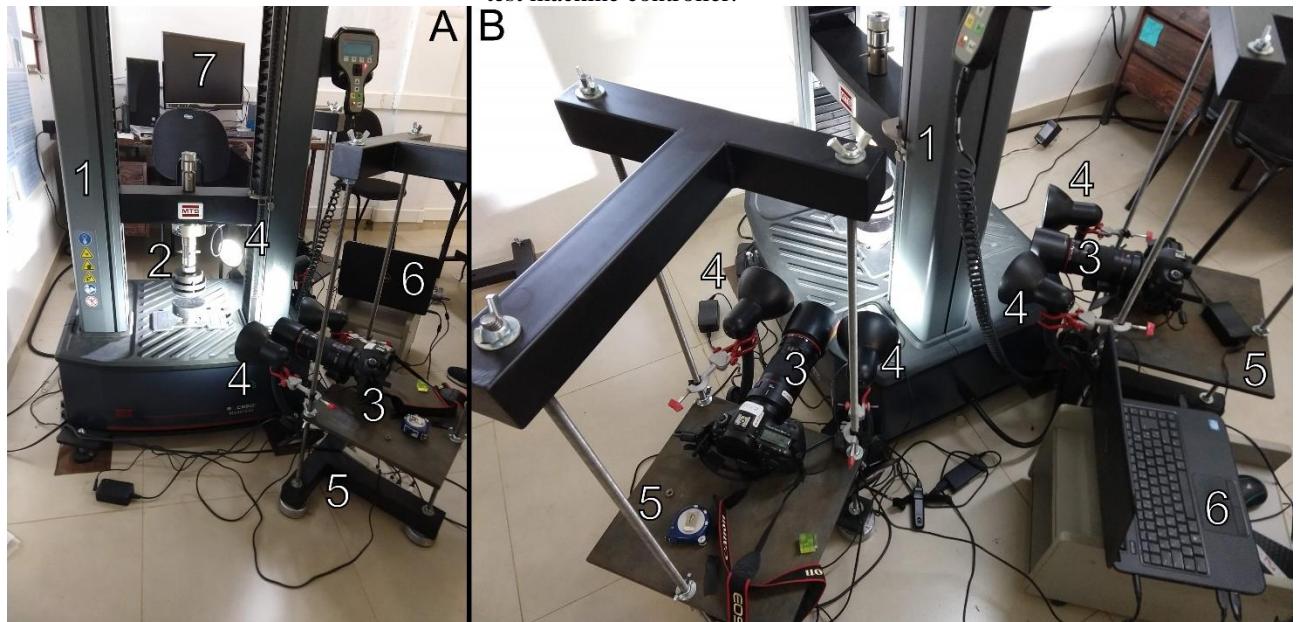


Figure 4 – Specimens with random speckle pattern positioned on the testing machine. A) Uniaxial compression specimen. B) A Diametral compression specimen.

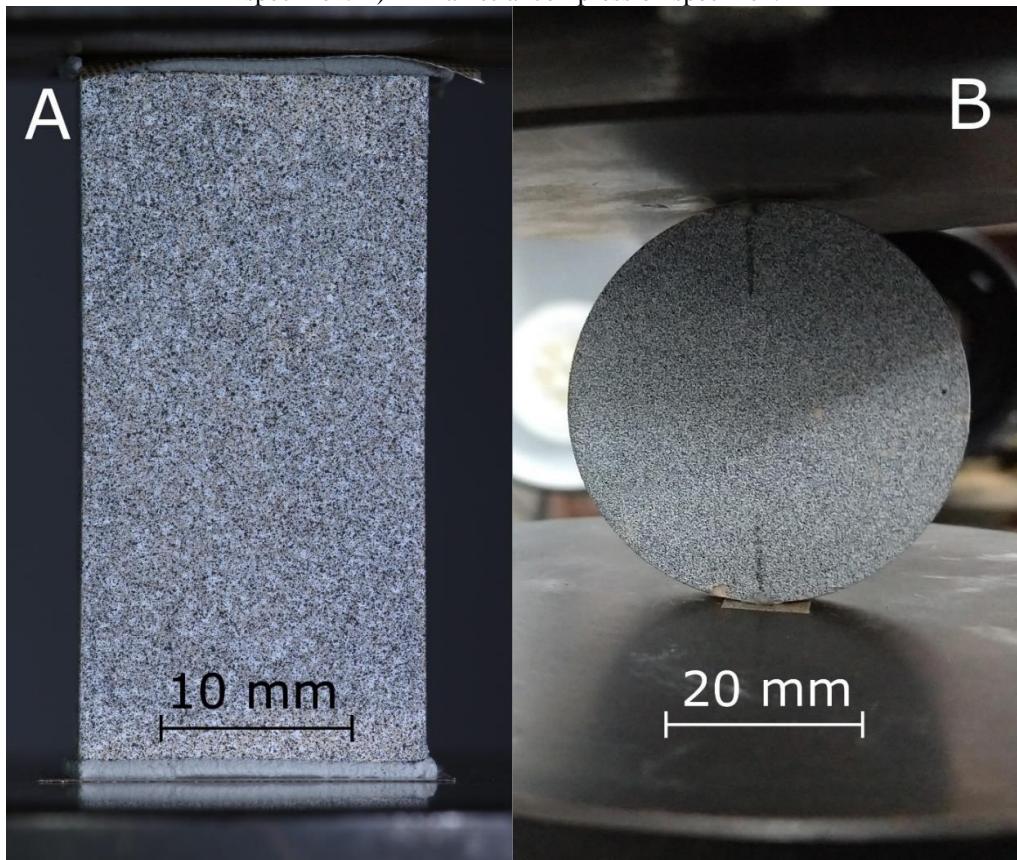


Figure 5 – Illustration of the arrangement of the cameras. A) Uniaxial compression test (90° angle). B) Diametral compression test (180° angle).

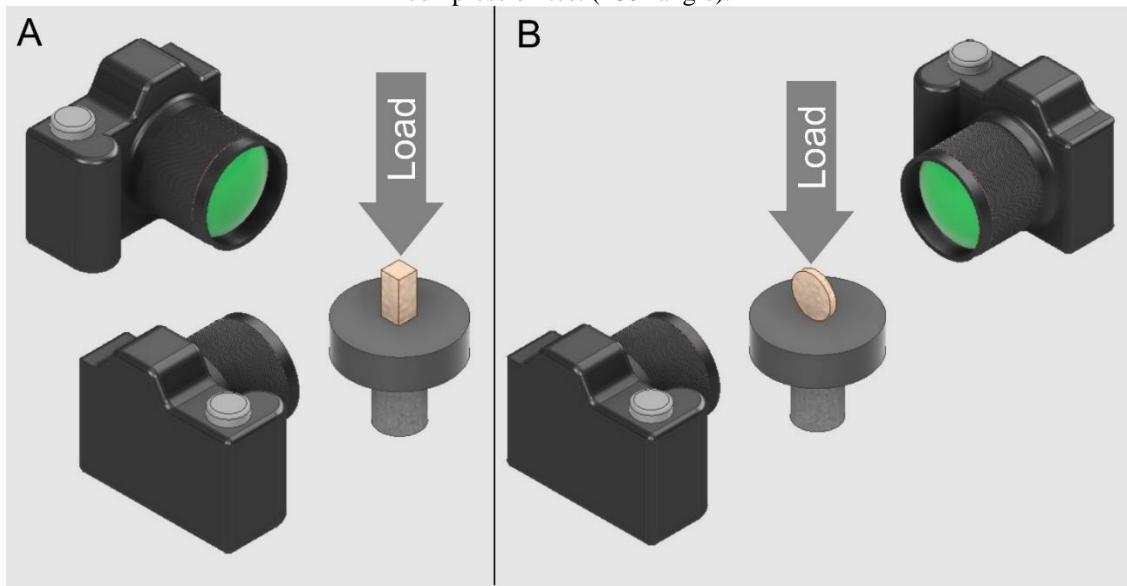


Figure 6 – Isostatic mold. A – Perspective view of a digital representation of the mold with section cut (the colors are only illustrative). B – Main dimensions of the mold. Dimensions in millimeters. C – Picture of the mold components and a pressed hemisphere.

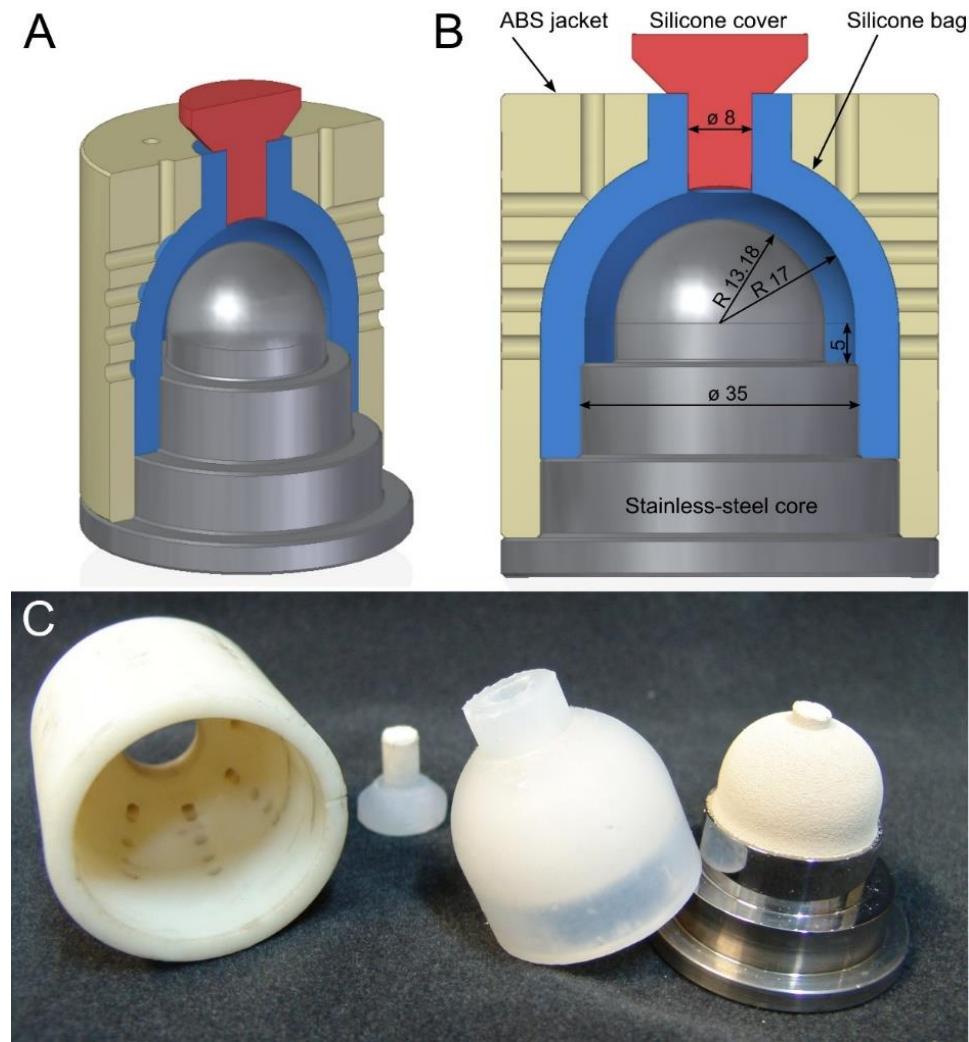


Figure 7 – Axisymmetric section of the FE model. Representation of the powder mesh with the main dimensions and with the loadings. Dimensions in millimeters. Symmetry axis on the left.

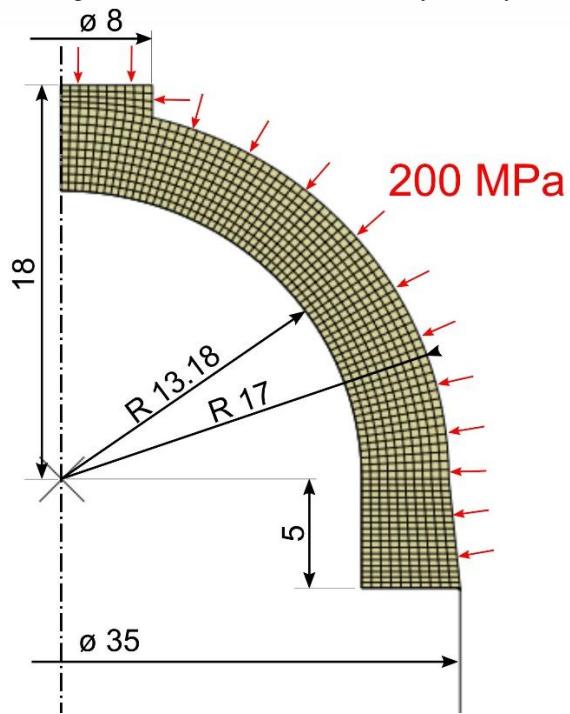


Figure 8 – Cycled uniaxial compression test of specimen U3. The first cycle presents negligible plastic deformation.

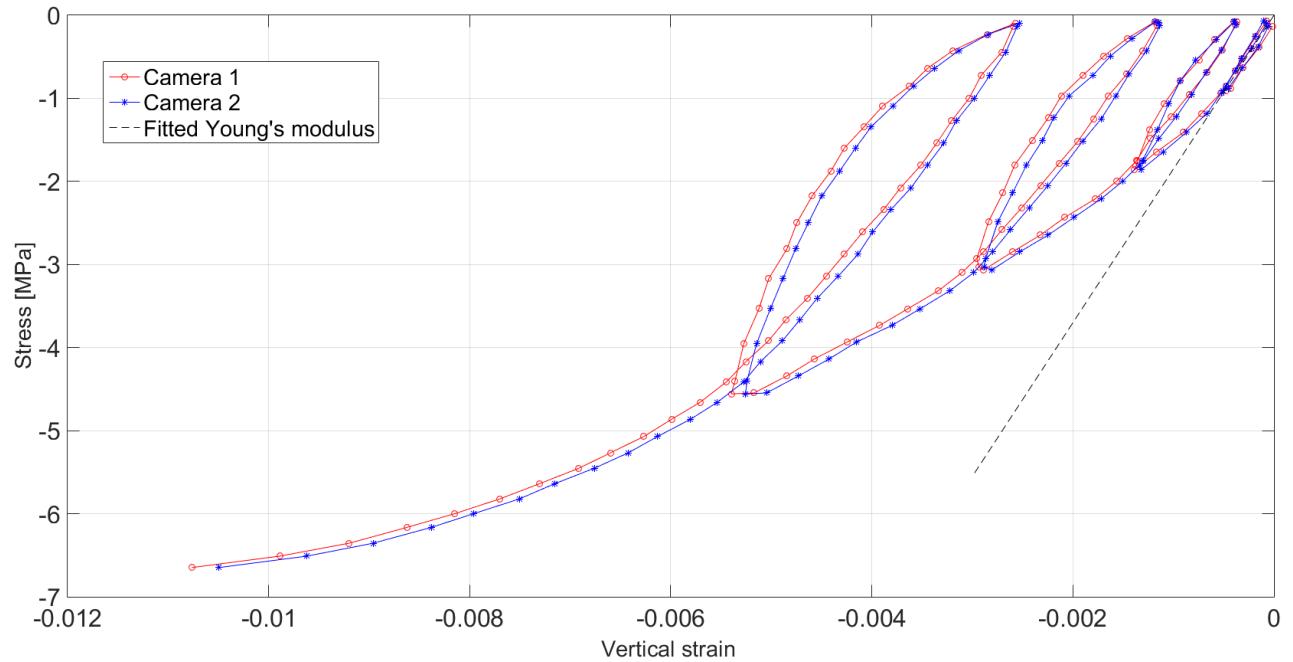


Figure 9 – Graph of the deviatoric strain by volumetric strain for uniaxial specimen U3. Arrows were added to highlight the beginning of the test and the dilatancy and fracture regions.

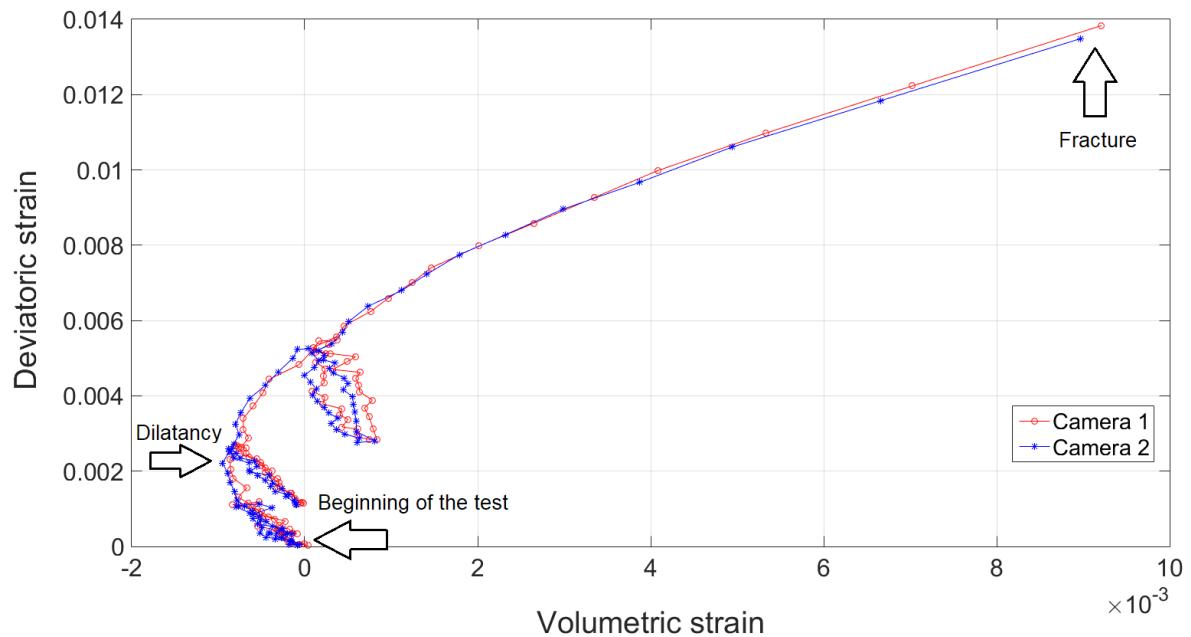


Figure 10 – Cycled diametral compression test of specimen D3 — average from cameras 1 and 2.

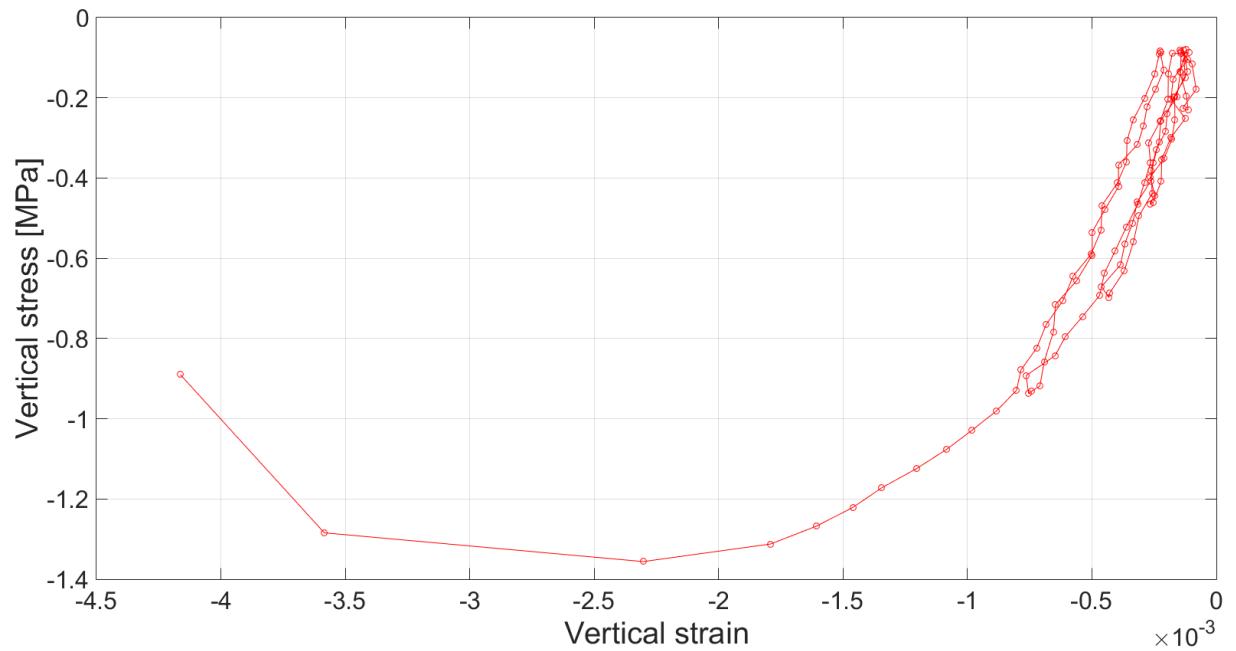


Figure 11 – Graph of the deviatoric strain by volumetric strain for diametral specimen D3 Average result from cameras 1 and 2. Arrows were added to highlight the dilatancy and fracture regions.

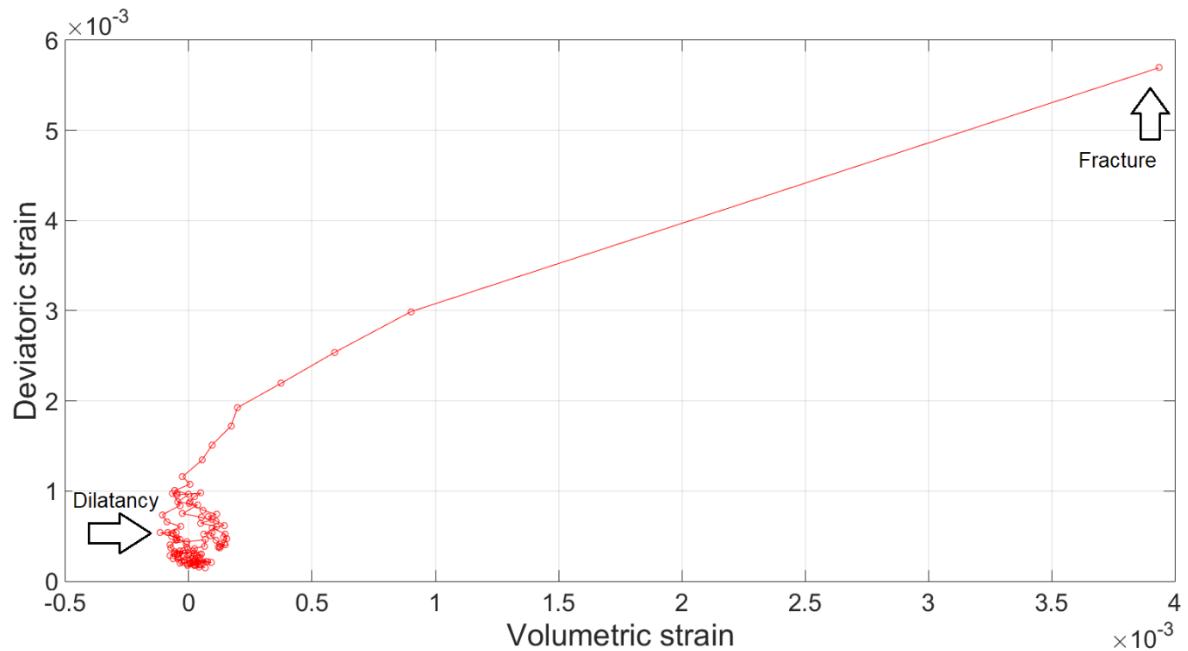


Figure 12 – Drucker-Prager failure surface adjusted to the uniaxial and diametral compression tests results, considering the dilatancy stress (blue) and fracture stress (red).

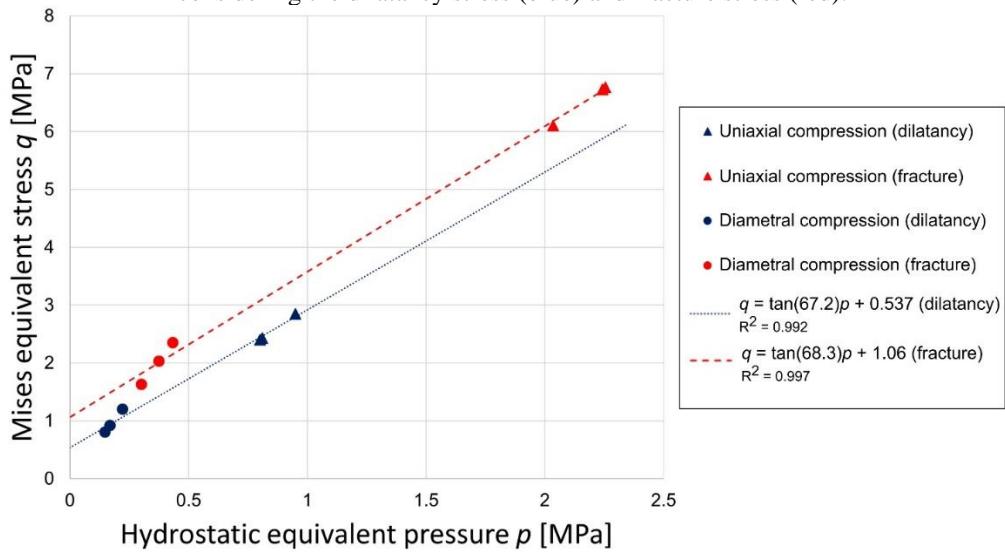


Figure 13 – Cap hardening fit.

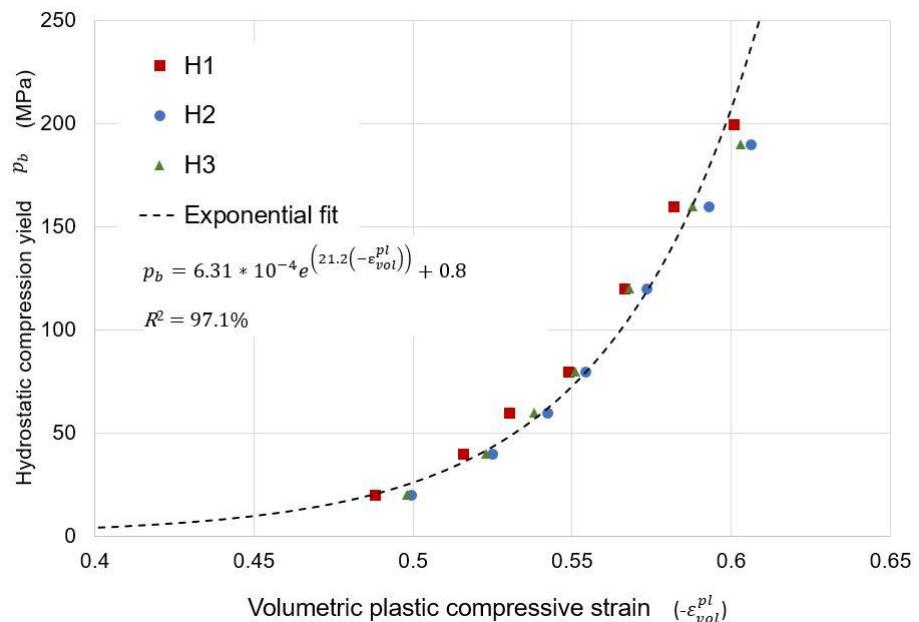


Figure 14 – Two compacted PZT hemispheres evidencing external and internal surfaces.



Figure 15 – On the left – density field resultant of the FE simulation with hardening and dilatancy. On the right – comparison between the measured profile and the simulations with and without hardening.

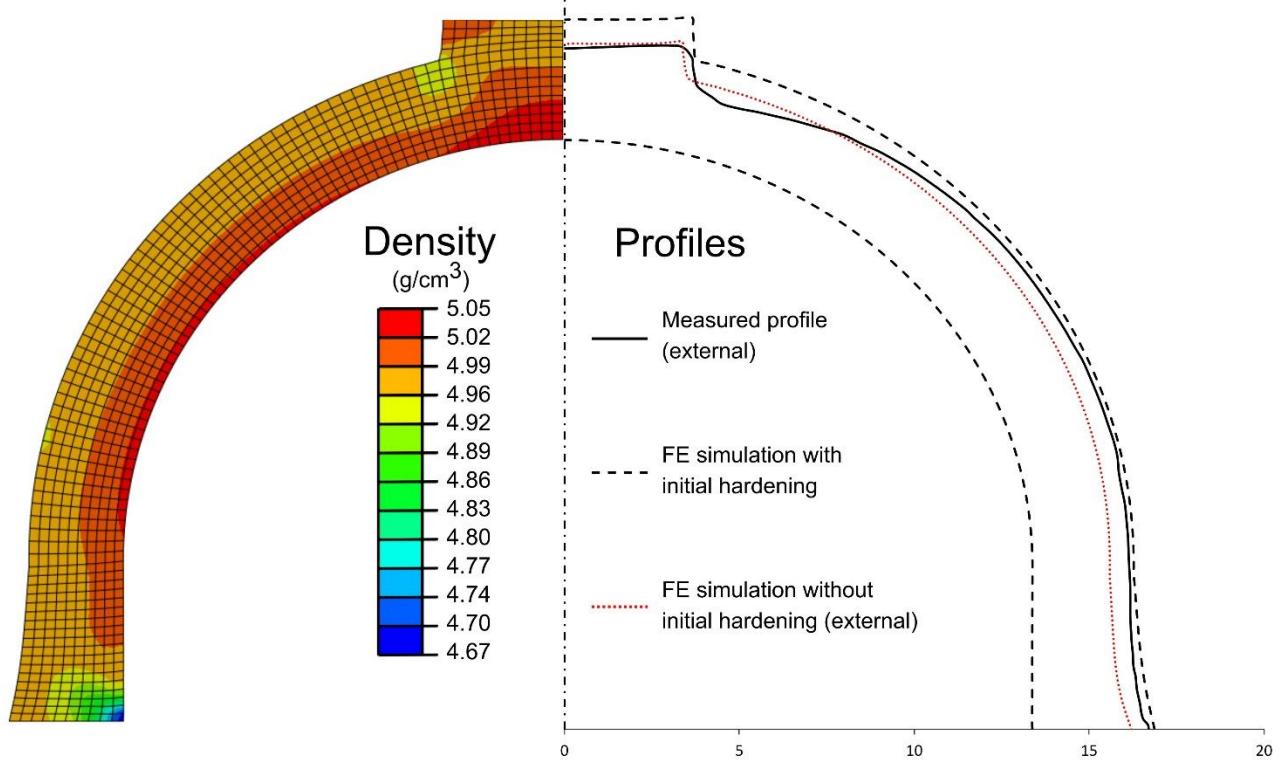


Figure 16 – Detail of the bottom of the part shaped as “elephant foot.” A – simulated density field. B – projected profile of the part (white dashed line added to highlight profile).

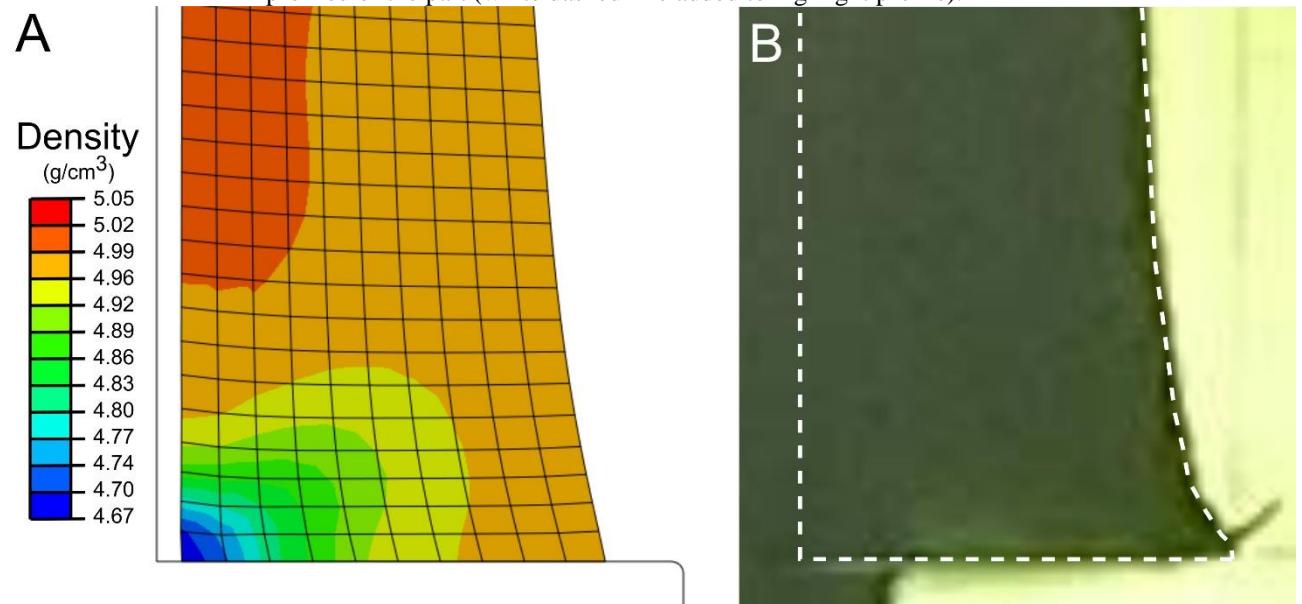
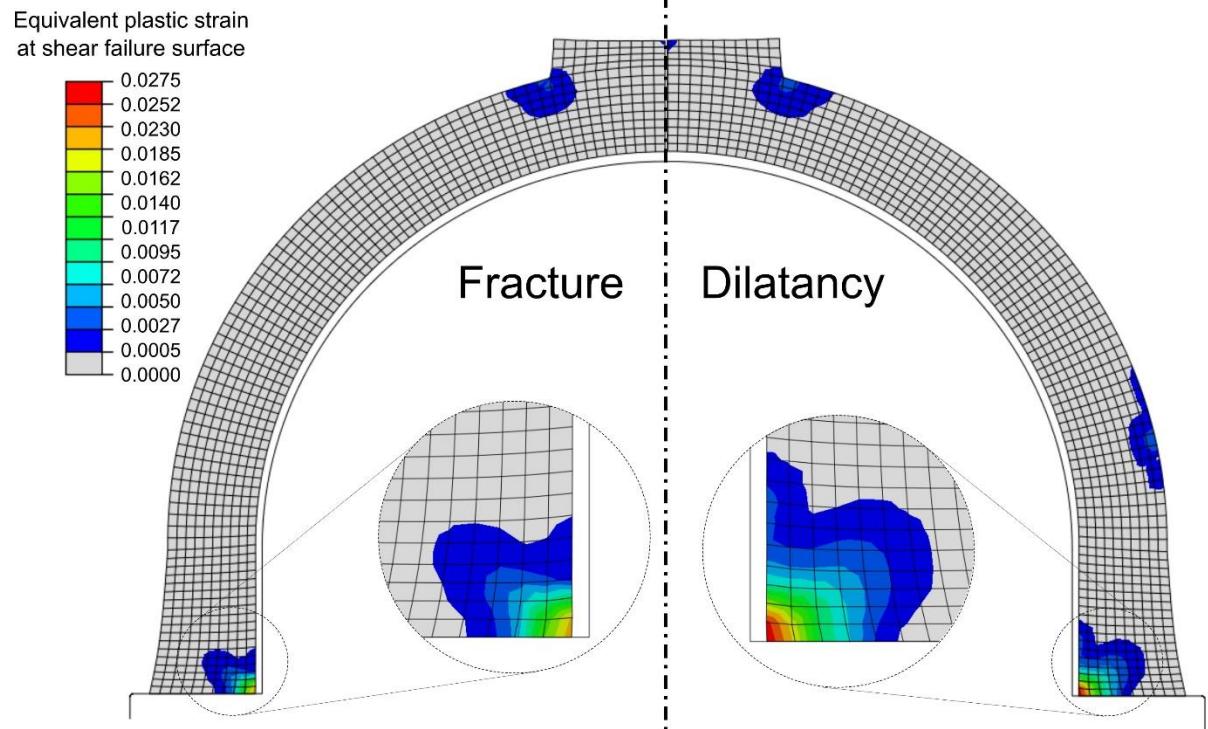


Figure 17 – Plastic strain caused by shear failure. On the left – simulation with parameters calibrated using fracture data. On the right – simulation with dilatancy data. The bottom region was magnified for improved comparison.



## SUPPLEMENTARY MATERIAL S1-PZT-DilatancyHardening-Ext

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 681, 691, 701, 711, 721, 731, 741, 751, 761, 861, 862, 863, 864, 865, 866, 867  
 868, 869, 870  
 \*Nset, nset=Symmetry  
 5, 7, 12, 160, 161, 162, 213, 214, 215, 216, 217, 218, 219, 220, 221  
 \*Elset, elset=Symmetry

551, 561, 571, 581, 780, 790, 800, 810, 820, 830, 840, 850, 860, 870  
 \*Nset, nset=ExternalSet  
 1, 4, 6, 7, 8, 11, 85, 86, 87, 88, 89, 90, 91, 92, 93, 94  
 95, 96, 97, 98, 99, 100, 101, 102, 103, 104, 105, 106, 107, 108, 109, 110  
 111, 112, 113, 114, 115, 116, 117, 118, 119, 120, 121, 122, 123, 124, 125, 126  
 127, 128, 129, 130, 131, 132, 133, 134, 135, 136, 137, 138, 148, 149, 150, 151  
 152, 153, 154, 155, 156, 157, 158, 159, 184, 185, 186, 187, 188, 189, 190, 191  
 192, 193, 194, 195, 196, 197, 198, 199  
 \*Elset, elset=ExternalSet  
 1, 11, 21, 31, 41, 51, 61, 71, 81, 91, 101, 111, 121, 131, 141, 151  
 161, 171, 181, 191, 201, 211, 221, 231, 241, 251, 261, 271, 281, 291, 301, 311  
 321, 331, 341, 351, 361, 371, 381, 391, 401, 411, 421, 431, 441, 451, 461, 471  
 481, 491, 501, 511, 521, 531, 541, 560, 570, 580, 581, 582, 583, 584, 585, 586  
 587, 588, 589, 590, 591, 601, 611, 621, 640, 650, 660, 670, 680, 690, 700, 710  
 720, 730, 740, 750, 760, 770  
 \*Elset, elset=\_SurfInt\_S2, internal  
 10, 20, 30, 40, 50, 60, 70, 80, 90, 100, 110, 120, 130, 140, 150, 160  
 170, 180, 190, 200, 210, 220, 230, 240, 250, 260, 270, 280, 290, 300, 310, 320  
 330, 340, 350, 360, 370, 380, 390, 400, 410, 420, 430, 440, 450, 460, 470, 480  
 490, 500, 510, 520, 530, 540, 550, 600, 610, 620, 630, 780, 790, 800, 810, 820  
 830, 840, 850, 860, 870  
 \*Elset, elset=\_SurfInt\_S3, internal  
 621, 622, 623, 624, 625, 626, 627, 628, 629, 630, 861, 862, 863, 864, 865, 866  
 867, 868, 869, 870  
 \*Elset, elset=\_SurfInt\_S4, internal, generate  
 591, 761, 10  
 \*Surface, type=ELEMENT, name=SurfInt  
 \_SurfInt\_S2, S2  
 \_SurfInt\_S3, S3  
 \_SurfInt\_S4, S4  
 \*Elset, elset=\_SurfExt\_S4, internal  
 1, 11, 21, 31, 41, 51, 61, 71, 81, 91, 101, 111, 121, 131, 141, 151  
 161, 171, 181, 191, 201, 211, 221, 231, 241, 251, 261, 271, 281, 291, 301, 311  
 321, 331, 341, 351, 361, 371, 381, 391, 401, 411, 421, 431, 441, 451, 461, 471  
 481, 491, 501, 511, 521, 531, 541, 591, 601, 611, 621  
 \*Elset, elset=\_SurfExt\_S2, internal  
 560, 570, 580, 590, 640, 650, 660, 670, 680, 690, 700, 710, 720, 730, 740, 750  
 760, 770  
 \*Elset, elset=\_SurfExt\_S3, internal, generate  
 581, 590, 1  
 \*Surface, type=ELEMENT, name=SurfExt  
 \_SurfExt\_S4, S4  
 \_SurfExt\_S2, S2  
 \_SurfExt\_S3, S3  
 \*\* Section: PZT\_Dilatancy  
 \*Solid Section, elset=HemisphereSection, material="PZT Dilatancy"  
 ,  
 \*End Part  
 \*\*  
 \*Part, name=MetalCore  
 \*End Part  
 \*\*

```

**
** ASSEMBLY
**
*Assembly, name=Assembly
**
*Instance, name=MetalCore-1, part=MetalCore
    0., -0.12, 0.
*Nset, nset=MetalCore-1-RefPt_, internal
1,
*Nset, nset="Reference core"
1,
*Surface, type=SEGMENTS, name=SurfCore
START, 0., 0.13318
CIRCL, 0.01318, 0.12, 0., 0.12
LINE, 0.01318, 0.115
LINE, 0.0174000070747003, 0.115
CIRCL, 0.0175, 0.1149000070747, 0.0174, 0.1149
LINE, 0.0175, 0.103
LINE, 0.0225, 0.103
LINE, 0.0225, 0.093
LINE, 0.0275, 0.093
LINE, 0.0275, 0.088
LINE, 0., 0.088
*Rigid Body, ref node=MetalCore-1-RefPt_, analytical surface=SurfCore
*End Instance
**
*Instance, name=Hemisphere-1, part=Hemisphere
*End Instance
**
*End Assembly
**
** MATERIALS
**
**
**
*Material, name="PZT Dilatancy"
*Cap Plasticity
537000., 67.21, 0.75, 0.297, 0.01, 1.
*Cap Hardening
    800631., 0.
    801817., 0.05
    805233., 0.1
    815074., 0.15
    843422., 0.2
    925081., 0.25
    1.16031e+06, 0.3
    1.83789e+06, 0.35
    3.78975e+06, 0.4
    4.49429e+06, 0.41
    5.36486e+06, 0.42

```

```

6.44058e+06, 0.43
7.76979e+06, 0.44
9.41224e+06, 0.45
1.14417e+07, 0.46
1.39495e+07, 0.47
1.70482e+07, 0.48
2.08771e+07, 0.49
2.56083e+07, 0.5
3.14545e+07, 0.51
3.86783e+07, 0.52
4.76044e+07, 0.53
5.86339e+07, 0.54
7.22626e+07, 0.55
8.91029e+07, 0.56
1.09912e+08, 0.57
1.35624e+08, 0.58
1.67396e+08, 0.59
2.06654e+08, 0.6
2.55164e+08, 0.61
*Density
2754.,
*Elastic
1.758e+09, 0.178
**
**
*Material, name="PZT Fracture"
*Cap Plasticity
1.06e+06, 68.3, 0.75, 0.295, 0.01, 1.
*Cap Hardening
800631., 0.
801817., 0.05
805233., 0.1
815074., 0.15
843422., 0.2
925081., 0.25
1.16031e+06, 0.3
1.83789e+06, 0.35
3.78975e+06, 0.4
4.49429e+06, 0.41
5.36486e+06, 0.42
6.44058e+06, 0.43
7.76979e+06, 0.44
9.41224e+06, 0.45
1.14417e+07, 0.46
1.39495e+07, 0.47
1.70482e+07, 0.48
2.08771e+07, 0.49
2.56083e+07, 0.5
3.14545e+07, 0.51
3.86783e+07, 0.52
4.76044e+07, 0.53
5.86339e+07, 0.54

```

```

7.22626e+07, 0.55
8.91029e+07, 0.56
1.09912e+08, 0.57
1.35624e+08, 0.58
1.67396e+08, 0.59
2.06654e+08, 0.6
2.55164e+08, 0.61
*Density
2754.,
*Elastic
1.758e+09, 0.178
**
** INTERACTION PROPERTIES
**
*Surface Interaction, name="Friction 03"
1.,
*Friction, slip tolerance=0.005
0.3,
*Surface Behavior, pressure-overclosure=HARD
*Surface Interaction, name="No friction"
1.,
*Friction
0.,
*Surface Behavior, pressure-overclosure=HARD
*Surface Interaction, name="No slip"
1.,
*Friction, rough
*Surface Behavior, pressure-overclosure=HARD
**
** BOUNDARY CONDITIONS
**
** Name: FixedReference Type: Displacement/Rotation
*Boundary
MetalCore-1."Reference core", 1, 1
MetalCore-1."Reference core", 2, 2
MetalCore-1."Reference core", 6, 6
** Name: Symmetry Type: Symmetry/Antisymmetry/Encastre
*Boundary
Hemisphere-1.Symmetry, XSYMM
**
** INTERACTIONS
**
** Interaction: Core - powder
*Contact Pair, interaction="Friction 03", type=SURFACE TO SURFACE, tracking=STATE, adjust=0.0
Hemisphere-1.Surflnt, MetalCore-1.SurfCore
** -----
**
** STEP: Loading
**
*Step, name=Loading, nlgeom=YES, inc=100000, unsymm=YES
*Static
0.0005, 1., 1e-07, 0.05

```

```

**
** LOADS
**
** Name: Pressure Type: Pressure
*Dsload
Hemisphere-1.SurfExt, P, 2e+08
**
** OUTPUT REQUESTS
**
*Restart, write, frequency=0
**
** FIELD OUTPUT: ExternalCoord
**
*Output, field, frequency=99999
*Node Output, nset=Hemisphere-1.ExternalSet
COORD,
**
** FIELD OUTPUT: FieldOutputs
**
*Output, field, time interval=0.05, time marks=NO
*Node Output
CF, RF, U
*Element Output, directions=YES
EVOL, LE, PE, PEEQ, PEMAG, PEQC, S
*Contact Output
CDISP, CSTRESS
**
** HISTORY OUTPUT: H-Output-1
**
*Output, history, variable=PRESELECT
*End Step
** -----
**
** STEP: Unloading
**
*Step, name=Unloading, nlgeom=YES, inc=100000, unsymm=YES
*Static
0.05, 0.97, 1e-07, 0.05
**
** LOADS
**
** Name: Pressure Type: Pressure
*Dsload
Hemisphere-1.SurfExt, P, 6e+06
**
** OUTPUT REQUESTS
**
*Restart, write, frequency=0
**
** FIELD OUTPUT: ExternalCoord
**
*Output, field, frequency=99999

```

```

*Node Output, nset=Hemisphere-1.ExternalSet
COORD,
**
** FIELD OUTPUT: FieldOutputs
**
*Output, field, time interval=0.05, time marks=NO
*Node Output
CF, RF, U
*Element Output, directions=YES
EVOL, LE, PE, PEEQ, PEMAG, PEQC, S
*Contact Output
CDISP, CSTRESS
**
** HISTORY OUTPUT: H-Output-1
**
*Output, history, variable=PRESELECT
*End Step
** -----
**
** STEP: Final unloading
**
*Step, name="Final unloading", nlgeom=YES, inc=10000, unsymm=YES
*Static
0.0001, 0.027, 1e-10, 0.001
**
** LOADS
**
** Name: Pressure Type: Pressure
*Dsload
Hemisphere-1.SurfExt, P, 600000.
**
** OUTPUT REQUESTS
**
*Restart, write, frequency=0
**
** FIELD OUTPUT: ExternalCoord
**
*Output, field, frequency=99999
*Node Output, nset=Hemisphere-1.ExternalSet
COORD,
**
** FIELD OUTPUT: FieldOutputs
**
*Output, field, time interval=0.005, time marks=NO
*Node Output
CF, RF, U
*Element Output, directions=YES
EVOL, LE, PE, PEEQ, PEMAG, PEQC, S
*Contact Output
CDISP, CSTRESS
**
** HISTORY OUTPUT: H-Output-1

```

\*\*

\*Output, history, variable=PRESELECT

\*End Step

## SUPPLEMENTARY MATERIAL S2-macroField\_DensityEVOL

```
from abaqusConstants import *
from odbAccess import *

# Instructions

# Change only the header with the name of the .odb file and the bulk density of the powder.

# Run this script in Abaqus CAE. Click in the menu File > Run script. Then select this file.

# The code will start doing the calculations for each frame. The progress will be shown at the bottom
# of the Abaqus interface.

# After calculation is finished. Choose the created variable in the field variables list (where standard
# variables are shown: displacement, strain, stress...)

# ****
# Modify the next variables accordingly:

odbPath = "PZT-S1-DilatancyHardening-Ext.odb"                      # path to output database
initialDensity = 3.70                                                 # initial density of the part
# ****

if odbPath in session.odbs.keys():
    session.odbs[odbPath].close()

odb = session.openOdb(name=odbPath,readOnly=False)
```

```

allSteps = session.odbData[odbPath].steps.keys()

for i in range(len(allSteps)):

    step = odb.steps[allSteps[i]]

    allFrames = session.odbData[odbPath].steps[allSteps[i]].frames.keys()

    if i==0:

        volumeinitial = step.frames[0].fieldOutputs['EVOL']

        for j in range(len(allFrames)):

            frame = step.frames[j]

            volume = frame.fieldOutputs['EVOL']

            DEFVOL = log(volume/volumeinitial)

            Density = initialDensity*(volumeinitial/volume)

            newFieldDEFVOL = frame.FieldOutput(name='DEFVOL', description='Plastic vol
strain', field=DEFVOL)

            newFieldDensity = frame.FieldOutput(name='Density', description='Density',
field=Density)

            print 'stepName = ', allSteps[i], ' frameNumber = ', allFrames[j]

    odb.save()

    odb.close()

    odb = session.openOdb(name=odbPath)

    session.Viewport(name='Viewport: 1', origin=(0.0, 0.0), width=192.5,
height=88.0)

    session.viewports['Viewport: 1'].makeCurrent()

    session.viewports['Viewport: 1'].maximize()

    session.viewports['Viewport: 1'].setValues(displayedObject=odb)

```