

# Optimized Sigmoid-Based Complete Ensemble Empirical Mode Decomposition for Energy Management in Hybrid Electric Vehicles

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**ABSTRACT** This paper proposes a sigmoid-based particle swarm optimization (PSO) for a complete ensemble empirical mode decomposition (SPSO-CEEMD) applied to the energy management system (EMS) of a hybrid electric vehicle (HEV). The low-frequency power demand, to be supplied by the lithium-ion battery (LIB) and internal combustion engine (ICE), is calculated by the CEEMD, while sigmoid functions define the ICE reference, avoiding discontinuities in the control strategy and limiting the response frequency in the implementation of power, velocity and angle control loops for the ICE butterfly valve actuator. High-frequency demand is handled by the ultracapacitor (UC), which controls the dc-link voltage. The sigmoid functions are optimized to reduce the ICE fuel consumption and the LIB aging, considering ICE emissions as constraints in the PSO. To make the UC available in next peak demands, its terminal voltage restoration is relaxed by a phase-lag compensator (PLC) tuned to actuate only after power delivers, which reduces the influence in the LIB dynamic. Experimental and numerical results under the HWYCOL, SC03 and a Brazilian real-world drive cycles show that SPSO-CEEMD reduces the total operational cost, LIB stress and aging compared to state-of-the-art strategies. Despite larger UC voltage restoration error with the PLC, LIB power dynamic is not significantly affected, increasing its lifetime by 2.74% and 10.96% compared to traditional PI and low-pass filter strategies, respectively. Moreover, the total operational cost is reduced by 18.28% and 47.54% in relation to the exclusive operation strategy (XOS) and interval type-2 fuzzy logic control (IT2-FLC) adapted from the literature.

**INDEX TERMS** Hybrid electric vehicle, sigmoid functions, energy management system, empirical mode decomposition, voltage restoration.

## NOMENCLATURE

### Abbreviations

ANN	artificial neural network	EMD	empirical mode decomposition
BBC	bidirectional boost converter	EMS	energy management system
CEEMD	complete ensemble empirical mode decomposition	EoL	end-of-life
CO	carbon monoxide	FC	fuel cell
NO <sub>x</sub>	nitrous oxides	FLC	fuzzy logic controller
DoD	depth of discharge	GA	genetic algorithm
		HEV	hybrid electric vehicle
		HIL	hardware-in-the-loop

HPTS	hysteresis power threshold strategy
ICE	internal combustion engine
ICEV	internal combustion engine vehicle
IMF	intrinsic mode function
IT2	interval type-2
LIB	lithium-ion battery
LBS	learning-based strategy
LPF	low-pass filter
LUT	lookup table
OBS	optimization-based strategy
PBC	passivity-based control
PE	permutation entropy
PEV	pure electric vehicle
PI	proportional-integral
PLC	phase-lag compensator
PMSM	permanent magnetic synchronous machine
PFCS	power follower control strategy
P-HEV	parallel hybrid electric vehicle
PMP	Pontryagins minimum principle
PWM	pulse-width modulation
PSO	particle swarm optimization
RBS	rule-based strategy
RL	reinforcement learning
S-HEV	series hybrid electric vehicle
SoC	state-of-charge
SoH	state-of-health
SSS	start-stop system
TCS	thermostat control strategy
UC	ultracapacitor
WT	Wavelet transform
XOS	exclusive operation strategy

#### Variables and constants

$v_b, i_b$	LIB voltage and current
$E_a$	LIB activation energy
$z, c$	LIB power-law factor and c-rate
$Ah$	total Ampere-hour throughput
$v_{oc}$	LIB open circuit voltage
$C_b, E_b$	LIB capacity and energy
$\Lambda$	LIB pre-exponential factor
$SoC_b$	LIB SoC
$DoD_b$	LIB DoD
$SoC_{min}$	LIB minimum SoC
$SoC_{max}$	LIB maximum SoC
$SoH_b$	LIB SoH
$SoH_b^{end}$	LIB final SoH
$\beta_{LIB}$	LIB pack price
$v_u, i_u$	UC voltage and current
$\Delta v_{uc}$	UC restoration control input
$i_{uc}^*, v_{uc}^*$	UC reference current and reference voltage
$F_t$	traction force
$M_v, A_v$	vehicle total mass and front area
$g$	gravitational acceleration
$R$	ideal gas constant

$v_x, \bar{v}_x$	vehicle speed and average speed
$c_r, c_k$	rolling resistance and slip stiffness
$f_F, k_R$	friction coefficient and tire slip
$\rho_a, \rho_f$	air and fuel density
$C_d$	aerodynamic coefficient
$P_d$	instantaneous power demand
$\mathcal{P}[n]$	HEV power demand sequence
$\omega_h, \mathcal{T}_h$	high-speed shaft angular velocity and torque
$\omega_l, \mathcal{T}_l$	low-speed shaft angular velocity and torque
$K_{sh}, B_{sh}$	high-speed shaft stiffness and damping
$K_{sl}, B_{sl}$	low-speed shaft stiffness and damping
$N_{rdl}, N_{rdh}$	low-speed and high-speed shafts output speed
$r_d$	wheel radius
$r_L$	load tire radio
$J_r, \eta_r$	speed reducer inertia and efficiency
$J_m$	traction motor inertia
$B_m$	traction motor viscous friction coefficient
$P_e, \eta_e$	ICE power and efficiency
$\dot{m}_f$	ICE fuel consumption
$P_e^{max}$	ICE maximum power
$b_e$	ICE fuel consumption rate
$\gamma_\mu$	ICE power coefficients
$P_g, \eta_g$	generator power and efficiency
$\omega_e, \mathcal{T}_e$	ICE angular velocity and torque
$\omega_g, \mathcal{T}_g$	generator angular velocity and torque
$\omega_n$	ICE normalized angular velocity
$P_o$	ICE low-rotation reference power
$\omega_{pp}$	angular velocity for maximum power
$\theta, \tau_v$	butterfly valve angle and delay
$J_e, J_g$	ICE and generator inertia
$T_f$	ambient temperature
$T_c, T_s$	LIB core and surface temperature
$T_a, T_a^{end}$	LIB internal temperature and its final value
$\mathcal{S}_{low}, \mathcal{S}_{high}$	low and high demand sigmoid functions
$\kappa$	sigmoid steepness
$\alpha$	power sharing coefficient
$c_1, c_2,$	PSO acceleration coefficients
$N_p$	PSO number of particles
Cost	total operational cost
$\Upsilon_m, \Upsilon_s$	PSO weights
$P_{LF}$	low-frequency power demand
$P_{bICE}^*$	LIB and ICE reference power
$P_{ICE}^*$	ICE reference power
$P_{ICE}^{max}$	ICE maximum power
$P_b^*$	LIB reference power
$w^h, \tau$	white noise and PE time delay
$K_m, K_c$	dc motor and chopper PWM gains
$m, v$	PE embedding dimension and vector
$\xi$	normalized PE
$P_o^A, P_o^B$	control parameters
$L_d, \phi$	road distance and slope
$A_w, D_w$	low and high-frequency wavelet components
$\sigma_w$	high-frequency component standard deviation
$\beta_f, \beta_b$	fuel cost and LIB energy cost
$IMF_k, r_k$	$k$ -th IMF and its residue



## I. INTRODUCTION

THE automotive industry has long faced challenges related to environmental pollution, primarily due to greenhouse gas emissions from internal combustion engine vehicles (ICEVs) and a heavy reliance on a singular energy source. The reduction of gas emissions has been extensively addressed in both academic research and legislative frameworks [1]. In response, the adoption of electric vehicles (EVs) and hybrid electric vehicles (HEVs) has gained widespread acceptance as potential solutions [2]–[4]. Among these, HEVs have emerged as the most widely adopted option, offering superior fuel efficiency compared to traditional ICEVs, while providing a longer cruising range than purely electric vehicles (PEVs) [5]. Moreover, the main pollutants as carbon monoxide (CO) and nitrous oxides (NO<sub>x</sub>) can be reduced [6].

To accommodate varying load dynamics, the powertrain architecture of HEVs has become increasingly diversified [2]. Ultracapacitors (UCs), with their high power density and extended life cycle, are utilized to supply and absorb high-frequency power demands. In contrast, low-frequency power demands are managed by other energy sources, such as batteries and internal combustion engines (ICEs) [3], [4]. Although the combination of a battery and UC as a hybrid storage system can enhance the HEV powertrain by integrating high specific energy storage with high specific power storage [3], many studies have not adopted this configuration. Even though UC control requires an additional dc-dc converter, its implementation enables each energy source to operate according to its dynamic response, thereby contributing to improve efficiency [4], [7]. However, managing multiple energy sources efficiently poses significant challenges, particularly concerning battery longevity and overall system performance.

For lithium-ion batteries (LIBs), one of the key performance indicators is the state-of-health (SoH), which is defined as the ratio between the current and initial capacity [8]. This metric is closely linked to LIB aging and its eventual end-of-life (EoL). Given that the battery is among the most expensive components of an EV [9], it is essential to monitor and control factors that contribute to its aging, such as temperature, charge and discharge currents, and depth-of-discharge (DoD) [9], [10]. Given these challenges, effective management of HEV energy sources is crucial. To this end, various EMS strategies have been proposed, which can be broadly classified into rule-based strategies (RBSs), optimization-based strategies (OBSs), and learning-based strategies (LBSs) [11], [12].

Due to their simplicity, low implementation complexity and ideal real-time performance [13], [14], various RBSs have been proposed in the literature. To reduce energy consumption, Phan et al. [15] introduced an interval type-2 fuzzy logic controller (IT2-FLC) to manage the uncertainties of driving conditions within an intelligent EMS

for parallel hybrid electric vehicles (P-HEVs). While this approach yields a higher final state-of-charge (SoC) and improved energy efficiency compared to traditional fuzzy logic controllers (FLCs), it does not account for battery SoH or thermal effects. Building on the thermostat control strategy (TCS) and power follower control strategy (PFCS), Shabbir and Evangelou's exclusive operation strategy (XOS) [16] enhanced the optimality of RBSs [17]. However, the number of ICE restarts in the ON-OFF operation mode is high, which can increase the pollutant emissions [18]. This is due to the fact that the emissions are reduced when the engine catalyst temperature reaches the light-off temperature [19], which is affected by start-stop system (SSSs). Furthermore, frequent engine restarts may generate noticeable noise and vibrations, potentially compromising passenger comfort.

Due to the limitations pointed out in the above mentioned approaches, alternative strategies incorporating neural network and signal processing techniques have been explored. Benmouna et al. [20] utilized an artificial neural network (ANN) in combination with passivity-based control (PBC) within the EMS. The ANN determines the battery reference current, while the PBC ensures compliance with this reference. Although this approach reduces battery stress, it leads to nearly constant current, suggesting underutilization of the energy source. Moreover, SoH is not included into this strategy, and the optimization of operating points for the energy sources is not addressed. Numerous other studies have employed WT to separate high and low-frequency power demands [21]–[23]. Nonetheless, the effectiveness of this method is strongly influenced by the choice of the base wavelet function and the number of decomposition levels [3].

Despite their low computational complexity and robust real-time performance, RBSs rely heavily on expert knowledge [14], [24]. Moreover, optimal results for fuel economy are not achieved [13]. To improve the RBSs performance, several works have combining them with OBSs. An optimized frequency decoupling strategy that combines a FLC with Wavelet transform (WT) is employed in [25], [26]. In this approach, the FLC determines the cut-off frequency of a low-pass filter (LPF), which is used to compute the UC reference, while WT handles power allocation based on the power demand frequency. However, the use of an LPF still permits some high-frequency demand to be achieved by the battery and the SoH is not considered. Additionally, the SoC reference for the UC is dependent on the vehicles speed. An optimized hysteresis power threshold strategy (HPTS) with an energy-based SSS for fuel optimization, including a fuel penalty for engine starts, was presented in [17]. The power reference for each source is defined according a 2-D map constructed according to an optimum analytical solution. To avoid excessive ICE state transitions, an hysteresis switching scheme is adopted. Moreover, Campos et al. [4] combined a RBS with offline optimization to reduce both fuel consumption and battery usage. However, battery aging and pollutant

emissions were not considered in [4], [17].

Numerous works have employed LBSs in the EMS of HEVs. Tang et al. [13] proposed a strategy based on deep reinforcement learning (RL) combined with a rule-based SSS to enhance fuel efficiency. However, pollutant emissions, which are significantly influenced by the SSS, were neglected. Similarly, Wang et al. [24] proposed the merging of Bayesian optimization with deep RL to improve robustness and reduce dependence on real-world data. Although energy consumption is reduced by the strategy, battery thermal model and aging were not considered, nor does it account for ICE emissions or actuator dynamics. A data driven RL method is proposed in [14] to the EMS in a HEV to improve the learning from a static offline dataset and the fine-tuning process while minimizes the fuel consumption. Battery aging, ICE emissions an actuator were overlooked. As highlighted by Hong et al. [19], most studies focusing on fuel consumption minimization tend to result in increased harmful gas emissions, demonstrating a typical trade-off between fuel savings and emissions.

Despite the optimal performance, LBSs require a large training data [4], [12]. These strategies are commonly used in autonomous vehicles, but still suffer with low reliability and safety in this application [27]. On the other hand, online OBSs application faces the problem of requiring high computational effort and memory [4], which makes their application in the onboard computation units of HEVs unfeasible [17]. For these reasons, RBSs are more commonly used in commercial HEVs [4], [17].

Although the recent strategies have made significant progress in optimizing the utilization of energy sources, the negative impact of high-frequency demands on battery performance is frequently neglected. To allocate high and low-frequency demands among the sources without the need for predefined basis functions, as required in Wavelet and Fourier-based strategies, the empirical mode decomposition (EMD) technique has emerged as a promising alternative [3]. The original signal is adaptively decomposed by EMD based on its temporal characteristics.

A challenge of the EMD techniques is that discontinuous signals may be improperly decomposed due to modal aliasing [28]. This issue can be addressed using the complete ensemble empirical mode decomposition (CEEMD) strategy [29]. Shen et al. [3] applied the CEEMD combined with permutation entropy (PE) to reduce high-frequency demands on the battery in EVs. Nevertheless, this approach requires adaptation for HEVs. Although battery aging is mitigated, the strategy does not achieve an optimal operating point for each energy source and a detailed battery thermal model is not developed, which can affect the accuracy of SoH evaluation.

Considering the crucial role of the battery in EMSs, a pre-heating strategy is proposed in [30] for a P-HEV. The battery pre-heating time is defined by a particle swarm optimization (PSO) problem, while the EMS is governed by using the Pontryagin's minimum principle (PMP) to reduce

fuel consumption and battery aging. However, the ICE emissions and actuator were neglected. Zha et al. [12] developed a time-efficient battery temperature sensitive EMS for a S-HEV. The strategy reduces the fuel consumption and keeps the battery temperature in the thermal comfort zone. Nonetheless, the ICE emissions and actuator were not taken into account, and the battery aging was not directly considered.

While temperature management is crucial for optimizing battery performance, another significant factor impacting the system's efficiency is the performance of the UC. After the device delivers power during high-frequency transients, its terminal voltage drops, which may limit its ability to manage subsequent charging and discharging cycles [3]. Therefore, restoring the UC voltage is critical, and various strategies have been proposed in the literature. Bastos et al. [31] introduced a method where the output of a LPF is added to the current reference of the device. The time constant is tuned to delay voltage restoration relative to current peaks by narrowing the voltage restoration bandwidth. In [32], a proportional-integral (PI) controller processes the UC terminal voltage error, generating an additional term that is split between the battery and UC in a microgrid, though no specific design criteria for the controller were provided. In Shen et al. [3], a method based on FLC to maintain the UC SoC at an appropriate level is proposed. However, the development of FLC strategies relies on expert knowledge.

Regarding the ICE in HEV, the use of discontinuous strategies to manage the ICE can be prejudicial. For instance, when the ICE shifts between two operating points with a considerable power difference, the resulting discontinuity can cause undesirable high-frequency oscillations. Additionally, the butterfly valve may not respond effectively due to its slower dynamic characteristics, making necessary to use a faster valve, which can increase the costs. In this way, although sigmoid functions have been widely utilized in various fields, such as SoC equalization [33] and dc microgrid control [34], their application in ICE management has not been thoroughly explored. These nonlinear functions offer several advantages, including a reduced number of tuning parameters, adaptability to diverse operational conditions, and continuity, making them strong candidates for ICE management.

Despite the importance of considering the actuator dynamics, many authors neglect them in the management and control, focusing only on the power balance equation [2], [5], [12], [16], [17], [35]. However, in practical applications, neglecting actuator dynamics can significantly impact stability and performance, as time delays and mechanical characteristics can alter system behavior. To address such challenges in valve control, a dual-loop cascade control strategy is proposed in [36] for a P-HEV. The outer loop regulates the longitudinal speed of the P-HEV, setting the torque reference to be supplied by the engines. The torque reference for the electric motor is used in a lookup table (LUT) to determine the reference angle, which is then em-

**TABLE 1.** State-of-art works and proposed strategy: features comparison

Reference	Features					
	(i)	(ii)	(iii)	(iv)	(v)	(vi)
This paper	✓	✓	✓	✓	✓	✓
[2], [13]	×	✓	×	×	×	×
[3], [21], [23]	×	×	*	✓	*	✓
[25]	×	×	*	✓	*	×
[4]	×	✓	×	✓	×	✓
[17]	×	✓	×	✓	×	×
[5]	✓	✓	×	×	×	✓
[35]	✓	✓	×	×	×	×
[15], [16]	×	×	×	✓	×	×
[26]	×	✓	*	✓	*	✓
[20]	×	✓	*	×	*	✓
[30]	✓	✓	×	✓	×	×
[38]	✓	✓	*	×	*	×
[12]	✓	✓	×	✓	×	×
[14], [24]	×	✓	×	×	×	×

\*Not applicable. Powertrain using fuel cell or other sources instead of ICE.

ployed in an outer angle control loop. However, LUT-based methods have the disadvantage of relying on predefined conditions established by pre-constructed maps [37], which do not account for real-time variations or uncertainties in engine behavior.

The above literature review highlighted the key attributes of each proposal with respect to the features: (i) inclusion of battery thermal effects and aging model in the EMS; (ii) optimization of fuel consumption and/or LIB aging; (iii) consideration of ICE emissions in the management/optimization; (iv) low computational complexity and no need for training data; (v) incorporation of the dynamic of the ICE actuator in both EMS and control, and (vi) integration of the battery and UC in a fully active topology, wherein both sources are controllable via a dc-dc converters. The comparison in relation to the literature state-of-art works is exhibited in Table 1.

This paper proposes a combined approach using CEEMD, PE and sigmoid functions to address battery aging, fuel consumption and ICE emissions in a S-HEV. For the best of authors' knowledge, this is the first application of the CEEMD and PE approach in the EMS of a HEV. The main contributions are summarized as follows:

- 1) Effectively mitigates high-frequency demands on the LIB and ICE without requiring basis functions, as needed in traditional WT strategies;
- 2) Avoid discontinuities that could cause high-frequency oscillations, and accommodates the slow dynamics of the ICE butterfly valve actuator by using sigmoid functions in the ICE management and a triple control loop strategy;
- 3) Ensures that UC voltage restoration occurs after the power peaks, minimizing any significant impact on the dynamic response of the other energy sources;
- 4) Balances the fuel consumption and LIB aging by optimizing the parameters of sigmoid functions and the demand distribution between the LIB and ICE

while simultaneously consider the ICE emissions as constraints in a optimization problem.

The rest of this paper is organized as follows. Section II presents the HEV powertrain structure and sources modeling. In Section III, the proposed SPSO-CEEMD strategy is discussed. Later, in Section IV, numerical and experimental results are shown, while the conclusion and future works are in Section V.

*Notation:* The sets  $\mathbb{N}$ ,  $\mathbb{R}_+$  and  $\mathbb{R}_+^*$  denotes the set of natural, non negative real and strictly positive real numbers respectively, and  $\text{sign}(x) = x/|x| \quad \forall x \neq 0$ .

## II. POWERTRAIN STRUCTURE AND MODELING

The series HEV (S-HEV) powertrain consists of a LIB, an UC, and an ICE coupled to a generator. Both the LIB and UC are connected to the dc-link via bidirectional boost converters (BBCs), controlled by pulse-width modulation (PWM) with duty-cycles  $d_b$  and  $d_{uc}$ , respectively, as shown in Fig. 1. The pulses  $p_b$  and  $\bar{p}_b$ , as well as  $p_{uc}$  and  $\bar{p}_{uc}$ , are complementary.

### A. VEHICULAR DYNAMIC

#### 1) Longitudinal dynamics

Considering the HEV energy consumption, the longitudinal dynamic is used to calculate the traction force as [30]:

$$F_t = M_v [\dot{v}_x + c_r g \cos(\phi) + g \sin(\phi)] + \frac{1}{2} \rho_a C_d A_v v_x^2 \quad (1)$$

where  $M_v$  is the vehicle total mass,  $C_d$  is the aerodynamic drag coefficient,  $A_v$  is the front area,  $\rho_a$  is the air density,  $\phi$  is the road slope,  $v_x$  is the longitudinal velocity and  $c_r$  is the rolling resistance, calculated as [39]:

$$c_r = \frac{r_L}{r_d} c_\kappa k_R + f_F - c_\kappa k_R \quad (2)$$

where  $r_d$  is the wheel radius,  $r_L$  is the load tire radius,  $c_\kappa$  is the tire normalized longitudinal slip stiffness,  $f_F$  is the friction coefficient and  $k_R$  is the tire slip, given by [40]:

$$k_R = \frac{\omega_l r_L - v_x}{v_x} \quad (3)$$

where  $\omega_l$  is the wheel angular velocity

*Remark 1:* Considering the free rolling tire model, the value of  $c_\kappa$  depends on  $k_R$ , which is neglected in this paper for simplification. Moreover, parasitic forces such as toe and camber, temperature, water and pressure effects are also neglected. All these influences can be analyzed in [40].

#### 2) Transmission unit

The transmission unit couples the electric motor, which operates at a high angular velocity  $\omega_h$ , to the vehicle wheels, rotating at a lower angular velocity  $\omega_l$ , through a gear reducer connecting the high-speed and low-speed shafts, as illustrated in Fig. 2a [41]. The dynamic models of both the high-speed and low-speed shafts from Matlab<sup>®</sup> were employed, with their respective torques computed as:

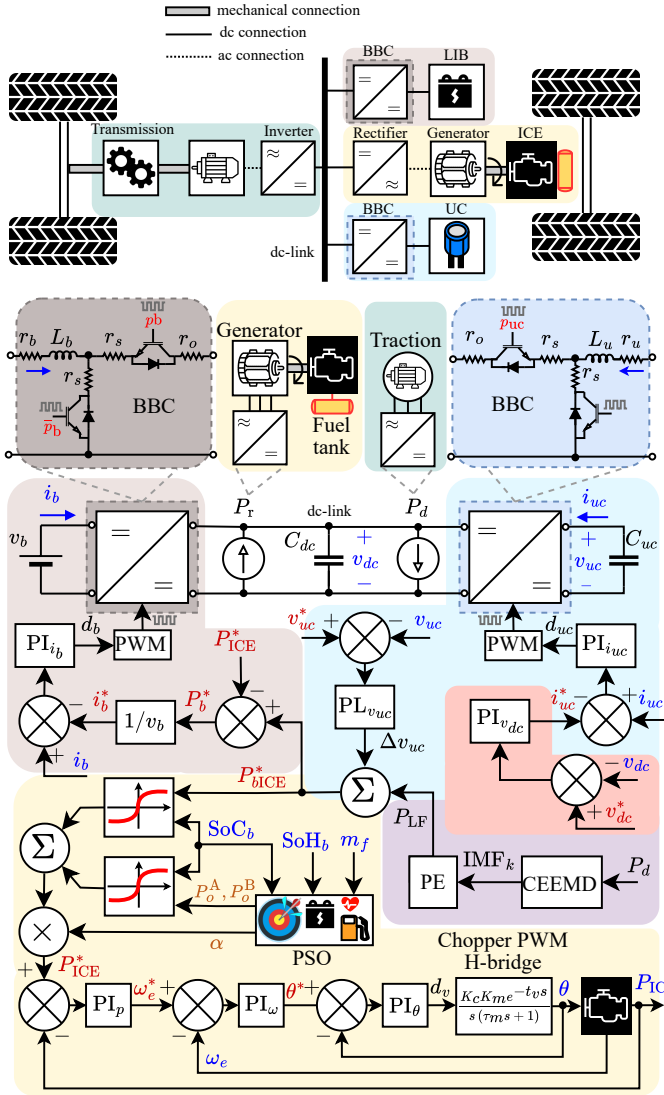


FIGURE 1. S-HEV powertrain and proposed SPSO-CEEMD EMS.

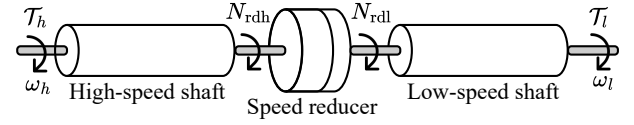
$$\dot{\tau}_h = K_{sh}(\omega_h - N_{rdh}) + B_{sh}(\dot{\omega}_h - \dot{N}_{rdh}) \quad (4a)$$

$$\dot{\tau}_l = K_{sl}(N_{rdl} - \omega_l) + B_{sl}(\dot{N}_{rdl} - \dot{\omega}_l) \quad (4b)$$

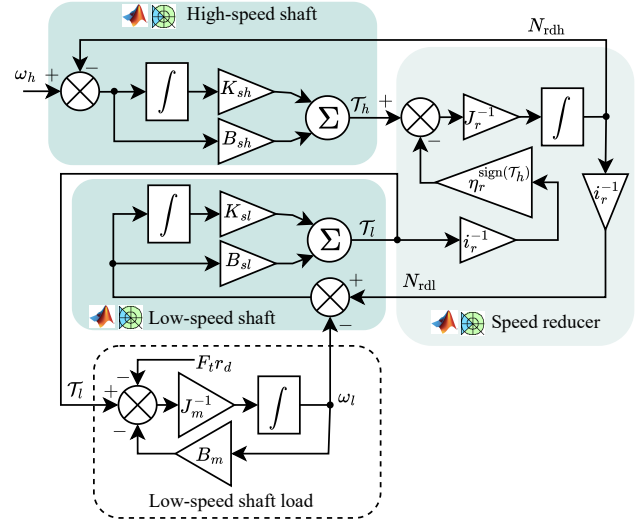
where  $N_{rdh}$  and  $N_{rdl}$  denote the output speeds of the high-speed and low-speed sides, respectively, while  $K_{sh}$  and  $B_{sh}$  represent the stiffness and damping of the high-speed shaft, and  $K_{sl}$  and  $B_{sl}$  correspond to the stiffness and damping of the low-speed shaft. The speed reducer is modeled according to [41]:

$$J_r \dot{N}_{rdh} = \tau_h - i_r^{-1} \eta_r^{\text{sign}(\tau_h)} \tau_l \quad (5)$$

with  $J_r$  and  $\eta_r$  representing the speed reducer inertia and efficiency, respectively,  $i_r \triangleq N_{rdh}/N_{rdl}$  corresponding to the reduction ratio [42], and  $\tau_l$  is the low-speed shaft torque.



(a)



(b)

FIGURE 2. HEV transmission unit: (a) Mechanical schematic [41] and (b) block diagram modeling.

The load coupled to the low-speed shaft is related to wheels inertia, mechanical resistances and the viscous friction coefficient of the motor, being calculated as [41], [42]:

$$J_m \dot{\omega}_l = \tau_l - F_t r_d - B_m \omega_l \quad (6)$$

where  $J_m$  is inertia of the vehicle's wheels concentrated on the rotor axis,  $r_d$  is the wheel radius, and  $B_m$  is the viscous friction coefficient of the motor. Therefore, the instantaneous power demand can be calculated as  $P_d = \tau_h \omega_h$ . The block diagram for the complete unit transmission is shown in Fig. 2b.

**Remark 2:** The electric motor traction control is not the focus of this work. Therefore, it is considered that the HEV reaches the desired velocity throughout the operation. Although machine control is not implemented in this work, it is widely studied in the literature and its application for a permanent magnetic synchronous machine (PMSM) can be seen in [41], [42].

## B. LITHIUM-ION BATTERY

The LIB model is crucial for representing its nonlinearities, which can influence the EMS. The adopted equivalent circuit model is the first-order RC, as shown in Fig. 3a. By applying



Kirchhoffs laws, the electrical model is defined as:

$$v_b = v_{oc} - R_{ob}i_b - v_{RC} \quad (7a)$$

$$\dot{v}_{RC} = (C_1 R_1)^{-1} (R_1 i_b - v_{RC}) \quad (7b)$$

where  $R_{ob}$  is the ohmic resistance,  $R_1$  and  $C_1$  are the polarization resistance and polarization capacitor,  $i_b$  and  $v_b$  are the LIB current and voltage, respectively, while  $v_{RC}$  is the terminal voltage on the RC circuit. The LIB resistances are higher for extremes SoC, as shown in Fig. 4. Therefore, it is recommended maintain SoC<sub>b</sub> within a certain level [43], once operating the LIB at extreme SoCs reduce its lifetime [33], [34]. The relationship between open-circuit voltage  $v_{oc}$  and SoC (estimated by the Coulomb's counting method) is given by the function  $f: \mathbb{R}_+ \rightarrow \mathbb{R}_+^*$ , shown in Fig. 5a.

#### 1) Aging model:

The LIB aging is related to its excessive usage, reaching a large number of cycles, which leads to a reduction of the capacity until reach the EoL [30]. This behavior can be measured by the SoH, calculated as [5]:

$$\dot{\text{SoH}}_b = -\frac{|i_b|}{2N_b C_b} \quad (8)$$

where  $N_b$  is the number of cycles and  $C_b$  is the LIB capacity. When the remaining available capacity reaches 20% of the nominal value, the EoL is reached and the total Ampere-hour  $A_h$  throughput can be calculated according to the Arrhenius' equation as [5]:

$$A_h = \left[ \frac{20}{\Lambda \exp(E_a/RT_a)} \right]^{1/z} \quad (9a)$$

$$E_a = 310.3c - 31,700 \quad (9b)$$

where  $R$  is the ideal gas constant,  $E_a$  is the activation energy,  $T_a$  is the LIB average temperature,  $z$  is the power-law factor and  $\Lambda$  is the pre-exponential factor, that depends on the LIB  $c$ -rate according to function  $g: \mathbb{R}_+ \rightarrow \mathbb{R}_+^*$  (see Fig. 5b). Therefore, the number of cycles  $N_b$  is given by [5], [30]:

$$N_b = 3,600 A_h / C_b \quad (10)$$

which allows evaluate the SoH in (8). As can be seen in (9a), the aging model is dependent on the temperature  $T_a$ , which requires a thermal model development.

#### 2) Thermal model:

Considering an longitudinal homogeneous cylindrical LIB, as shown in Fig. 3b [35], the energy conservation principle produces the thermal state-space model [38]:

$$\begin{bmatrix} \dot{T}_s \\ \dot{T}_c \end{bmatrix} = \begin{bmatrix} \frac{-(R_c + R_u)}{R_u R_c C_s} & \frac{1}{R_c C_s} \\ \frac{1}{R_c C_c} & \frac{1}{R_c C_c} \end{bmatrix} \begin{bmatrix} T_s \\ T_c \end{bmatrix} + \begin{bmatrix} \frac{1}{R_u C_s} & 0 \\ 0 & \frac{1}{C_c} \end{bmatrix} \begin{bmatrix} T_f \\ Q \end{bmatrix} \quad (11)$$

where  $T_s$ ,  $T_c$  and  $T_f$  are the LIB surface, core and ambient temperature, respectively,  $R_c$  and  $R_u$  are resistances that represent the heat conduction inside and at the surface, while  $C_c$  and  $C_s$  are the core and surface heat capacities. The

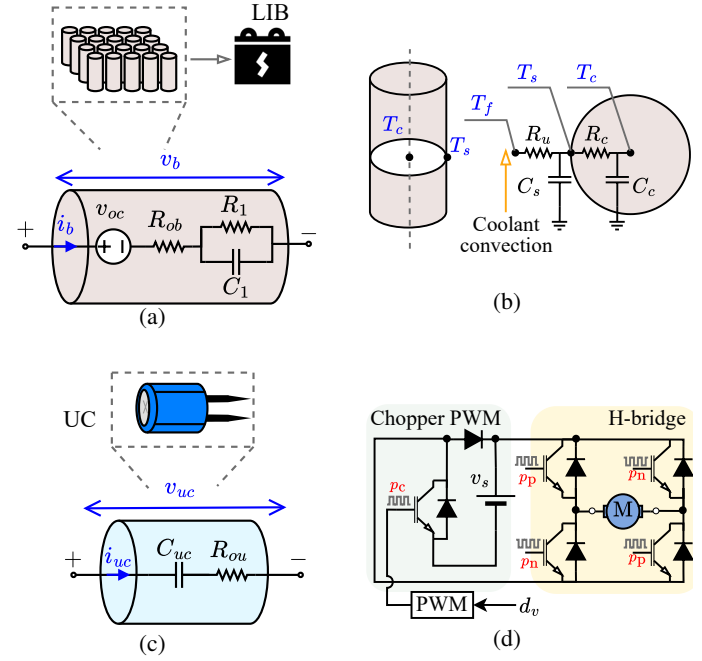


FIGURE 3. Sources modeling: (a) LIB electrical model, (b) LIB thermal model, (c) UC, and (d) ICE butterfly valve actuator.

internal temperature is given by  $T_a = 0.5(T_c + T_s)$  [35], [38], while the heat generation rate  $Q$  is calculated as:

$$Q = i_b (v_b - v_{oc}(\text{SoC}_b)) + i_b T_c \frac{\partial v_{oc}(\text{SoC}_b)}{\partial T_c}. \quad (12)$$

#### C. ULTRACAPACITOR

To precisely modeling the UC without a large computational cost, the equivalent circuit of an ideal capacitor  $C_{uc}$  with a series resistance  $R_{ou}$  was applied, shown in Fig. 3c, due to its simplicity. The UC SoC is proportional to its terminal voltage  $v_{uc}$  [3].

#### D. INTERNAL COMBUSTION ENGINE AND GENERATOR

The ICE actuator, shown in Fig. 3d, consists of a dc source  $v_s$  modulated by an H-bridge according the duty-cycle  $d_v$ . The H-bridge command the dc motor (M) rotation by the complementary pulses  $p_p$  and  $p_n$ , while the ICE dynamic is modeled by the third order polynomial approximation pre-

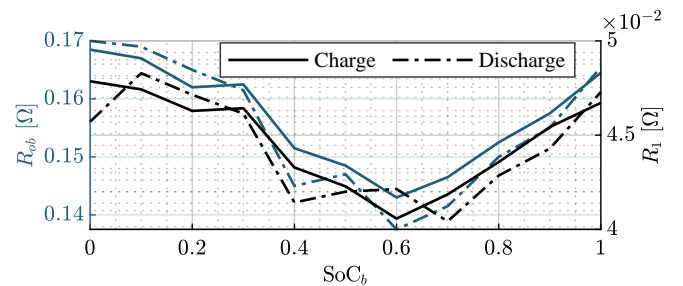
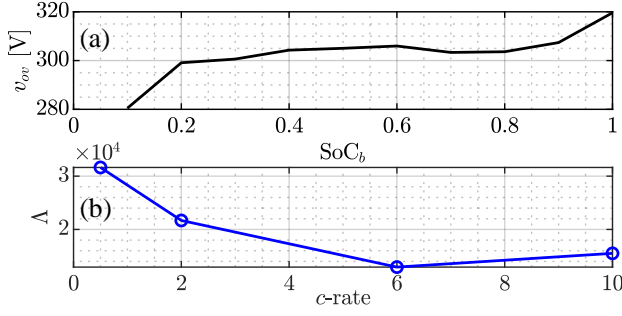


FIGURE 4. LIB resistances as a function of SoC.





**FIGURE 5.** Battery modeling: (a) LIB open circuit voltage as a function of SoC and (b) Pre-exponential factor  $\Lambda$  as a function of  $c$ -rate [5].

sented in [44]. Therefore, the ICE power can be calculated as:

$$P_e = \frac{P_e^{\max} \theta}{100} \sum_{\mu=1}^3 \gamma_{\mu} \omega_n^{\mu} \quad (13)$$

where  $\omega_n \triangleq \omega_e / \omega_{pp}$  with  $\omega_e$  the ICE angular velocity and  $\omega_{pp}$  the angular velocity for maximum power  $P_e^{\max}$ ,  $\gamma_{\mu} \in \mathbb{R}$  are coefficients dependent on the ICE used, with  $\mu \in \{1, 2, 3\}$ , and  $\theta \in [0, 100\%]$  is the butterfly valve angle. Moreover, the coupling between the ICE and generator is modeled by the rotational relationship [5]:

$$(r_{eg}^{-2} J_e + J_g) \dot{\omega}_g = \mathcal{T}_e r_{eg}^{-1} - \mathcal{T}_g \quad (14)$$

where  $J_e$  and  $J_g$  are the ICE and generator inertia,  $\omega_g$  is the generator angular velocity,  $\mathcal{T}_e$  and  $\mathcal{T}_g$  are the ICE and generator torque, respectively, and  $r_{eg} \triangleq \omega_g / \omega_e$ . The generator power  $P_g$  and the rectifier power  $P_r$  are given by:

$$P_g = \mathcal{T}_g \omega_g \eta_g \quad (15a)$$

$$P_r = P_g \eta_r \quad (15b)$$

where  $\eta_g$  and  $\eta_r$  are the generator and rectifier efficiencies, respectively.

#### 1) ICE fuel consumption map

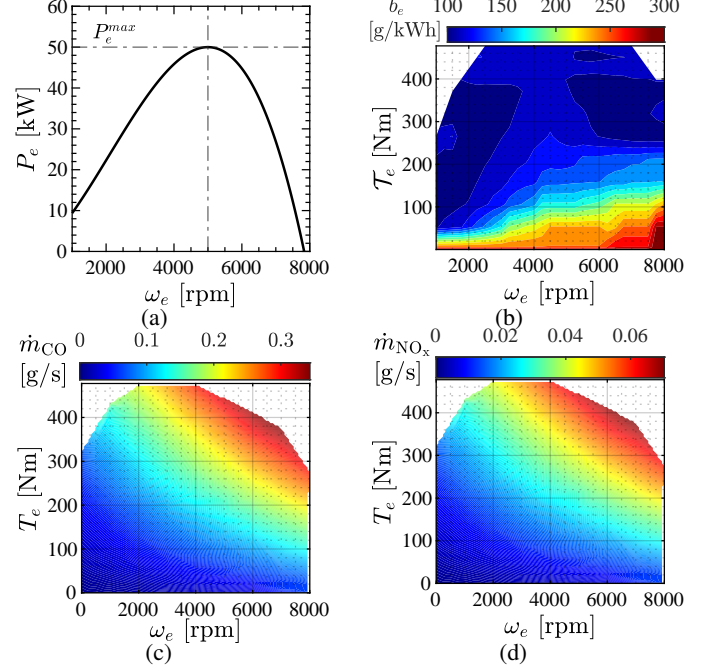
The ICE fuel consumption is calculated by a quasi-static model as [2]:

$$\dot{m}_f = \frac{\mathcal{T}_e \omega_e b_e}{367.1 \rho_f g} \quad (16)$$

where  $\rho_f$  is the fuel density and  $b_e$  is the fuel consumption rate. The ICE power map, dependent on the ICE angular velocity, and the fuel consumption rate [g/kWh], dependent on the ICE angular velocity and torque, are shown in Fig. 6a and 6b, respectively.

#### 2) ICE emission maps

The vehicle emission is affected by many factors such as engine type, driver, road characteristics among others [1]. For the sake of simplicity, the emission model in this paper is considering using static maps depending on the ICE velocity and torque. The CO and NO<sub>x</sub> emission rate are  $\dot{m}_{CO}$  [g/s] and  $\dot{m}_{NO_x}$  [g/s], respectively, and can be found in Fig. 6c and Fig. 6d.



**FIGURE 6.** ICE maps: (a) power, (b) fuel consumption ( $b_e$ ), (c) CO emission and (d) NO<sub>x</sub> emission.

### III. SIGMOID-CEEMD ENERGY MANAGEMENT

The proposed EMS is shown in Fig. 1. The low frequency demand, calculated by the CEEMD strategy, is shared between the ICE and the LIB according sigmoid functions optimized by the PSO algorithm. The UC controls the dc-link voltage while a phase-lag compensator (PLC) restores its terminal voltage.

#### A. POWER ALLOCATION BASED ON CEEMD AND PE

The EMD algorithm decomposes a signal into different time-scale characteristics without the need for predetermined harmonic orders or basis functions, as required in Fourier and WT, yielding a series of intrinsic mode functions (IMFs) [3]. The EMD method in [28] decomposes an original input signal by considering its local maxima and minima values through a shifting process. Initially, an upper envelope formed by the local maxima and a lower envelope formed by the local minima of the original signal are constructed. Therefore, the first component denoted  $\psi_1[n]$  is defined as:

$$\psi_1[n] = u[n] - m_1[n] \quad (17)$$

with  $u[n]$  the input signal and  $m_1[n]$  the average signal between these envelopes. The shifting is repeated  $p$  times, considering the component obtained from the previous process as the original signal:

$$\psi_{11}[n] = \psi_1[n] - m_{11}[n] \quad (18)$$

$$\psi_{1p}[n] = \psi_{1(p-1)}[n] - m_{1p}[n] \quad (19)$$

where  $m_{1p}$  is the  $p$ -th envelope used. Therefore, defining  $\text{IMF}_1[n] = \psi_{1p}[n]$  as the first IMF component, the first residue  $r_1[n]$  is calculated as:

$$r_1[n] = u[n] - \text{IMF}_1[n]. \quad (20)$$

The process is then repeated considering  $r_1[n]$  as the original signal until the standard deviation between two consecutive shifting results is less than a threshold. At the end of  $K$  repetitions, the same number of IMFs is obtained. *Definition 1:* The operator denoted  $\mathcal{E}_k(\cdot)$  extracts the  $k$ -th mode of the input signal, i.e., its  $k$ -th IMF using the EMD decomposition.

#### 1) Complete ensemble empirical mode decomposition

To suppress the modal aliasing effect without affect the signal properties, the CEEMD inserts white noises to the original signal  $u[n]$  [29]. Different realizations of white noise  $w^h[n]$ , of finite standard deviation  $\sigma \in (0, 1)$ , are added to the signal to obtain:

$$u^h[n] = u[n] + w^h[n] \quad (21)$$

with  $h = \{1, \dots, H\}$  and the first IMF calculated as:

$$\text{IMF}_1[n] = H^{-1} \sum_{h=1}^H \mathcal{E}_1(u^h[n]) \quad (22)$$

which allows to calculate the first residues and the second IMF by new decomposition as:

$$r_1[n] = u[n] - \text{IMF}_1[n] \quad (23)$$

$$\text{IMF}_2[n] = H^{-1} \sum_{h=1}^H \mathcal{E}_1(r_1[n] + \epsilon_1 \mathcal{E}_1(w^h[n])) \quad (24)$$

where  $\epsilon_1$  is the ratio of data versus noise. For  $k = \{1, \dots, K\}$  the  $k$ -th residue is calculated as  $r_k[n] = r_{(k-1)}[n] - \text{IMF}_k[n]$  and the  $(k+1)$ -th mode is [29]:

$$\text{IMF}_{(k+1)}[n] = H^{-1} \sum_{h=1}^H \mathcal{E}_1(r_k[n] + \epsilon_k \mathcal{E}_k(w^h[n])) \quad (25)$$

and the process is finished when the final residue  $r_K[n]$  is less than an established threshold, satisfying:

$$u[n] = r_K[n] + \sum_{k=1}^K \text{IMF}_k[n]. \quad (26)$$

Let  $\mathcal{P}[n] \in \mathbb{R}^{1 \times N}$ , with  $n = \{1, \dots, N\}$ , be the sequence of the HEV power demand  $P_d(t)$ , for  $t \in [0, t_{\max}]$ . Therefore, the CEEMD in this paper is applied considering  $u[n] = \mathcal{P}[n]$ .

#### 2) Permutation entropy

The application of the CEEMD algorithm in the HEV power demand produces a set of  $K$  IMFs, as described in (26). Each sequence  $\text{IMF}_k[n]$  presents a level of complexity that is related with its frequency response. Therefore, the high and low frequency IMF sequences are separated according to the PE.

Considering the embedding dimension  $2 \leq m \in \mathbb{N}$  and the time delay  $\tau$ , the power demand of the  $k$ -th IMF sequence  $\text{IMF}_k[n] = \{\text{imf}_k^n\}_{n=1}^N = \{\text{imf}_k^1, \text{imf}_k^2, \dots, \text{imf}_k^N\}$  can be written as the embedding vector [45]:

$$v = \{\text{imf}_k^1, \text{imf}_k^{1+\tau}, \dots, \text{imf}_k^{1+(m-1)\tau}\}. \quad (27)$$

Then, the  $m$  elements of the embedding vector are rearranged in ascending order [3], [45]:

$$\text{imf}_k^{n+(q_1-1)\tau} \leq \text{imf}_k^{n+(q_2-1)\tau} \leq \dots \leq \text{imf}_k^{n+(q_m-1)\tau} \quad (28)$$

where  $q_1, q_2, \dots, q_m$  is the column where each element is located, which allows mapping any embedding vector  $v$  into the space mapping vector  $\{q_1, q_2, \dots, q_m\} \in \mathbb{N}^m$  [45]. In a variety of arrangements, there are  $m$  spacing mapping vectors with probability of occurrence  $p_1, p_2, \dots, p_s$ , with  $s \leq m!$ . Therefore, the normalized PE is calculated as [3]:

$$\chi(m, \tau) = -\frac{\sum_{i=1}^s p_i(\tau) \ln(p_i(\tau))}{\ln(m!)}. \quad (29)$$

To analyze the  $k$ -th IMF frequency characteristics according to its PE denoted  $\chi_k$ , with  $k = \{1, \dots, K\}$  and calculated according (29), the larger the value of  $\chi_k$  the less regular is the sequence, i.e., IMFs with higher PE values are signals more complex, and then regarded as high frequency demand [3]. Therefore, the low frequency power demand and the reference power for the LIB and ICE are calculated as:

$$P_{\text{LF}} = \sum_k \text{IMF}_k[n] \quad / \quad \chi_k(\text{IMF}_k, \tau) \leq \sigma \quad (30)$$

$$P_{\text{bICE}}^* = P_{\text{LF}} + \Delta v_{uc} \quad (31)$$

where  $\Delta v_{uc}$  is a term for UC terminal voltage restoration, which will be discussed later.

#### B. ICE MANAGEMENT

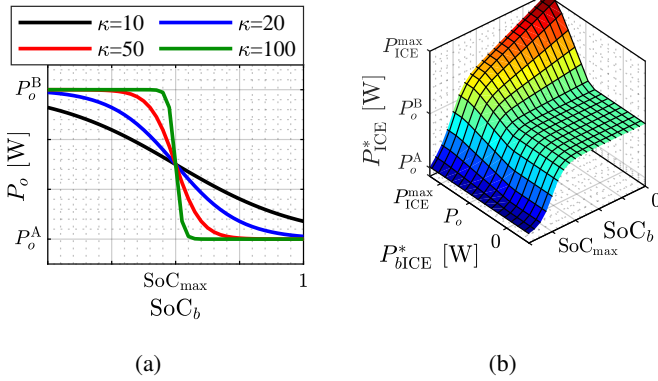
To avoid discontinuities in the EMS, which can lead to undesirable high frequency oscillations, the sigmoid functions are employed to calculate the ICE reference power as:

$$P_{\text{ICE}}^* = \alpha \left( P_o + \text{DoD}_b \underbrace{(\mathcal{S}_{\text{low}} + \mathcal{S}_{\text{high}})}_{\Delta P_{\text{ICE}}} \right) \quad (32)$$

where  $\text{DoD}_b \triangleq 1 - \text{SoC}_b$ ,  $\alpha \in [0, 1]$  is a gain to be defined and  $\mathcal{S}_{\text{low}}$  and  $\mathcal{S}_{\text{high}}$  are sigmoid functions to produce displacements around the low rotation reference power  $P_o$ , calculated as:

$$P_o = \frac{P_o^A}{1 + e^{-\kappa(\text{SoC}_b - \text{SoC}_{\max})}} + \frac{P_o^B}{1 + e^{\kappa(\text{SoC}_b - \text{SoC}_{\max})}} \quad (33)$$

where  $\kappa$  is the sigmoid steepness and  $P_o^A, P_o^B \in \mathbb{R}_+$  are design parameters so that  $P_o^A < P_o^B$ . When  $\text{SoC}_b \geq \text{SoC}_{\max}$  the lower rotation demand is defined as the minimum value  $P_o^A$ . However, if  $\text{SoC}_b \leq \text{SoC}_{\max}$ , the lower rotation demand is defined as  $P_o^B$  (see Fig. 7a).



**FIGURE 7.** ICE management: (a) low rotation reference power and (b) total reference power.

The sigmoid functions  $S_{low}$  and  $S_{high}$  in (32), used to calculate the displacement  $\Delta i_{ICE}$ , are given by:

$$S_{low} = \frac{P_{bICE}^* - P_o}{1 + e^{-\kappa_m(P_{bICE}^* - P_o)}} \frac{1}{1 + e^{-\kappa_m(P_{ICE}^{max} - P_{bICE}^*)}} \quad (34a)$$

$$S_{high} = \frac{(P_{ICE}^{max} - P_o)}{1 + e^{\kappa_m(P_{ICE}^{max} - P_{bICE}^*)}} \quad (34b)$$

where  $P_{ICE}^{max}$  is the ICE maximum power. When  $P_{bICE}^* \leq P_o$ , the ICE reference power is  $P_{ICE}^* = P_o^B$  when  $SoC_b \leq SoC_{max}$  (green plateau in Fig. 7b). Moreover, when  $P_o \leq P_{bICE}^* \leq P_{ICE}^{max}$ , the ICE reference power is approximately linear with respect to the demand  $P_{bICE}^*$ . Otherwise, if the demand is higher than  $P_{ICE}^{max}$ , the reference is limited to this value.

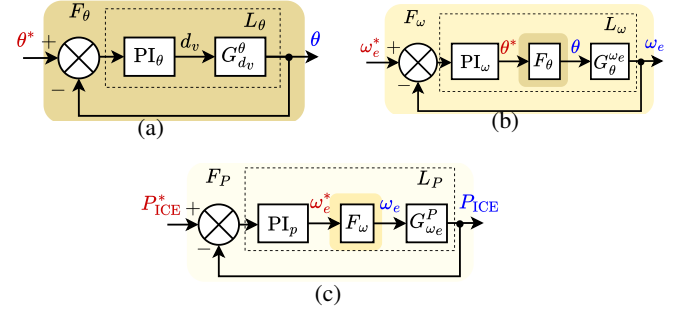
Once the ICE power reference is set, a triple control loop strategy is employed to determine the angular opening of the butterfly valve. The inner angular control loop (Fig. 8a) must be faster than the speed control loop (Fig. 8b) which, in turn, should be faster than the outer power control loop (Fig. 8c) to ensure effective control performance. This hierarchical control was achieved through the appropriate selection of the  $PI_p$ ,  $PI_\omega$  and  $PI_\theta$  controllers gains, ensuring that the open-loop cutoff frequency of the speed control loop ( $L_\omega$ ) is approximately 0.5 to 1 decade lower than that of the angular control loop ( $L_\theta$ ), while the open-loop cutoff frequency of the power control loop ( $L_P$ ) is approximately 0.5 to 1 decade lower than that of the speed control loop.

### C. LIB MANAGEMENT

Once the ICE reference power is defined, the LIB reference power  $P_b^*$  is calculated as:

$$P_b^* = P_{bICE}^* - P_{ICE}^* \quad (35)$$

**Remark 3:** Substituting (32) in (35) it is possible to see that the gain  $\alpha$  defines whether the ICE or LIB is more used to supply the lower frequency demand in (30). Higher values of  $\alpha$  result in greater use of the ICE, while lower  $\alpha$  result in greater use of the LIB.



**FIGURE 8.** ICE butterfly triple control loop: (a) angle control loop, (b) angular velocity control loop and (c) power control loop.

### D. ULTRACAPACITOR MANAGEMENT

The high frequency demand is designated as reference for the UC, due to its high power density, by using the dc-link voltage  $v_{dc}$ , its reference  $v_{dc}^*$  and a voltage control loop as:

$$i_{uc}^* = PI_{v_{dc}}(v_{dc}^* - v_{dc}) \quad (36)$$

where  $PI_{v_{dc}}$  is a PI controller with proportional and integral gains  $k_p^{dc}$  and  $k_i^{dc}$ , respectively.

#### 1) Voltage restoration

To restore the UC terminal voltage after power peaks, a PLC  $PL_{v_{uc}}$  is used. The PLC is chose to improve the stationary response without affect the transient response, i.e., to increase the gain at low frequencies without significantly affect the root locus in the vicinity of the closed-loop poles. Therefore, the voltage error is used to calculate the signal:

$$\Delta v_{uc} = \underbrace{\left( \frac{\beta_u \tau_u s + \beta_u}{\beta_u \tau_u s + 1} \right)}_{PL_{v_{uc}}} (v_{uc}^* - v_{uc}) \quad (37)$$

where  $\tau_u$  is the zero time constant and  $\beta_u$  is the low-frequency gain. The control signal  $\Delta v_{uc}$  is used in (31).

When compared to the traditional PI strategy, the PLC presents reduced low-frequency gain, which relaxes the voltage restoration. However, the phase delay does not affect significantly the transient response if the pole and zero are sufficiently close. In relation to the LPF, the PLC tuning method is improved due to the higher degree of freedom. Moreover, the LPF presents a trade-off between low-frequency gain and cut-off frequency.

The control signal for UC voltage restoration is incorporated as a reference for both the LIB and the ICE in (31). However, since the UC is a fast dynamic source, employing a PI controller may result in high-frequency demands on the LIB and ICE, which is undesirable. Therefore, to ensure that the PLC does not significantly affect the transient response of the controlled system, the pole ( $-1/\beta_u \tau_u$ ) and zero ( $-1/\tau_u$ ) in (37) must be positioned close to each other. Furthermore, the improvement in steady-state error is directly proportional to the ratio between the zero and the pole, i.e., the value of  $\beta_u$ . Therefore, to achieve a

substantial reduction in steady-state error while maintaining the transient response, the pole and zero of the PLC should be located near the origin.

### E. PARTICLE SWARM OPTIMIZATION

To optimize the HEV operation, the PSO, a population-based on the stochastic algorithm characterized by high versatility and flexibility [46], is applied. The PSO algorithm seeks to find the value  $x^* = [\alpha \ P_o^A \ P_o^B]$  that minimizes an objective function  $\mathcal{F}(x) : \mathbb{R}^d \rightarrow \mathbb{R}$  by introducing  $N_p$  particles into the search space with positions  $x_k \in \mathbb{R}^d$ , moving towards its personal best and global position with velocities  $v_k \in \mathbb{R}^d$ ,  $k = \{1, 2, \dots, N_p\}$ . Starting from the initial conditions  $x_k^0$  and  $v_k^0$ , the position and velocity of each particle are updated according to

$$\begin{cases} x_k^{n+1} = x_k^n + v_k^{n+1} \\ v_k^{n+1} = v_k^n + c_1 R_1^n (y_k^n - x_k^n) + c_2 R_2^n (\bar{y}_k^n - x_k^n) \end{cases} \quad (38)$$

where  $c_1, c_2 \in \mathbb{R}$  are acceleration coefficients,  $R_1^n, R_2^n \in [0 \ 1]$  are random numbers,  $y_k^n$  and  $\bar{y}_k^n$  are the best position of particle  $k$  and of all particles at iteration  $n$ , respectively, calculated as:

$$y_k^{n+1} = y_k^n + 0.5 (x_k^{n+1} - y_k^n) \mathbb{S}(x_k^{n+1}, y_k^n) \quad (39)$$

$$\bar{y}_k^{n+1} = \operatorname{argmin} \left\{ \mathcal{F}(y_1^{n+1}), \mathcal{F}(y_2^{n+1}), \dots, \mathcal{F}(y_{N_p}^{n+1}) \right\} \quad (40)$$

where the operator  $\mathbb{S}(a, b) \triangleq 1 + \operatorname{sign}[\mathcal{F}(b) - \mathcal{F}(a)]$  [46]. The PSO finds the optimum solution that minimizes the compromise among fuel consumption, LIB aging and SoC limits according to

$$\min_x \mathcal{F}(x) = \int_0^{t_{\max}} \Upsilon_m \dot{m}_f + \Upsilon_s \dot{S}oH_b \quad (41a)$$

$$\text{s.t. } SoC_{\min} \leq SoC_b \leq SoC_{\max} \quad (41b)$$

$$0 \leq \alpha \leq 1 \quad (41c)$$

$$0 \leq P_o^A \leq P_o^B \leq P_e^{\max} \quad (41d)$$

$$\bar{m}_{CO} < k_{CO} \bar{m}_{CO}^{\text{ICE}} \quad (41e)$$

$$\bar{m}_{NO_x} < k_{NO_x} \bar{m}_{NO_x}^{\text{ICE}} \quad (41f)$$

where  $\Upsilon_m$  and  $\Upsilon_s$  are compromised weights between fuel consumption and LIB aging to solve the multiobjective optimization,  $\bar{m}_{CO}$  and  $\bar{m}_{NO_x}$  are the CO and NO<sub>x</sub> emission rate [g/km], respectively, and  $\bar{m}_{CO}^{\text{ICE}} = 3.67$  g/km and  $\bar{m}_{NO_x}^{\text{ICE}} = 0.81$  g/km are the medium and heavy-duty ICE average emissions [6], while the gains  $k_{CO}, k_{NO_x} \in [0 \ 1]$  define the desired emission reduction, i.e., the higher the gains are the more relaxed the constraints become.

The influence of the PSO on the sigmoid functions is illustrated in Fig. 9. For a fixed value of  $P_o^B$ , as the PSO increases  $P_o^A$ , the difference between the maximum and minimum power of the ICE decreases, indicating that its operation occurs with reduced power variation (see Fig. 9a). Conversely, when  $P_o^A$  is fixed and  $P_o^B$  increases, the opposite behavior is observed (see Fig. 9b). Higher values of

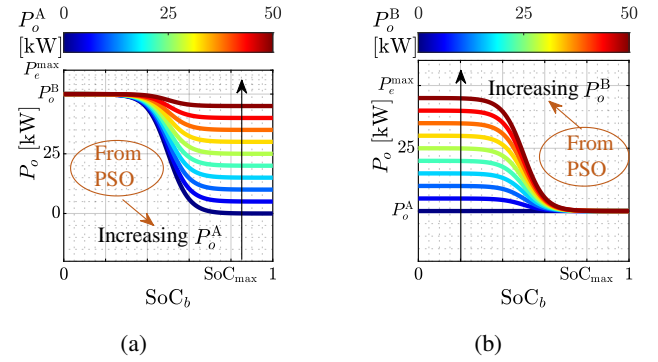


FIGURE 9. PSO influence in the ICE sigmoid functions: increasing (a)  $P_o^A$  and (b)  $P_o^B$ .

$P_o^A$  and  $P_o^B$  demand greater power from the ICE, thereby relieving the load on the LIB. Furthermore, since the gain  $\alpha$  multiplies both sigmoid functions in (32), higher  $\alpha$  values increase the difference between the ICE's operational points.

### IV. NUMERICAL AND EXPERIMENTAL RESULTS

In this section, the performance of the proposed sigmoid-based particle swarm optimization strategy is verified using numerical simulations and a real-time hardware-in-the-loop (HIL) platform composed of the Speedgoat™, where the S-HEV plant is built, and the dSPACE™, where the strategy is implemented, as shown in Fig. 10a.

The dSPACE1006 multiprocessor system, equipped with an AMD Opteron™ processor, acquires analog measurements from the Speedgoat via the 16-bit A/D board DS2004 and produces the PWM signals through the digital board DS4004. On the other hand, the S-HEV measurements are collected from Speedgoat (performance core Intel®Core™ i3 FPGA 100k) using the 16-bit analog output board IO110, while the PWM signals are received through the digital board IO316. The schematic of the board connections is shown in Fig. 10b.

The results were obtained considering three drive cycles with different characteristics regarding velocity, travel distance ( $L_d$ ), average velocity ( $\bar{v}_x$ ) and duration. The selected drive cycles were the SC03, HWYCOL and the Brazilian real-world drive cycle between the cities of Campinas and São Paulo (see Fig. 11a), collected and presented by Miranda et al. [39]. The speed profile of the drive cycles are shown in Fig. 11b and the emission constraints (41e) and (41f) are disregarded unless stated otherwise.

The LIB stress analysis is done by the power decomposition in low ( $A_w$ ) and high frequency ( $D_w$ ) components using the Haar Wavelet decomposition, described in [23]. The high frequency component standard deviation  $\sigma_w$ , around the null average value, indicates the utilization of the LIB [47]. The electrical and control parameter are described in Table 2 and Table 3. The total operational cost [\$/100 km] is calculated as:



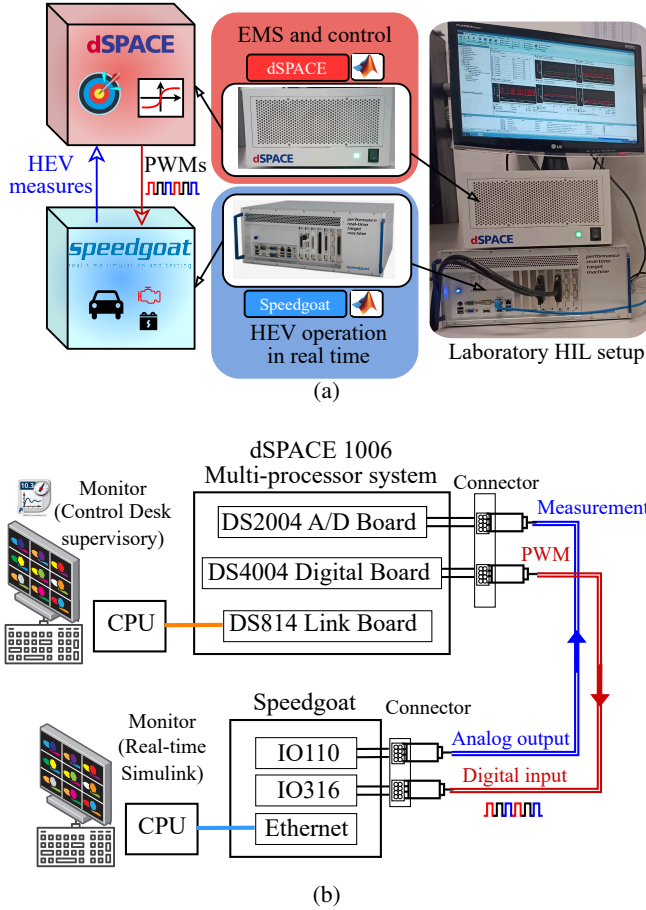


FIGURE 10. Real-time HIL setup: (a) Lab prototype and (b) Schematic of board connections.

$$\text{Cost} = \frac{100}{L_d} \left( E_b \beta_b + F_c \beta_f + \left( 1 - \text{SoH}_b^{\text{end}} \right) C_b \beta_{\text{LIB}} \right) \quad (42)$$

where  $E_b$  [kWh] is the LIB energy with associated cost  $\beta_b = 0.13$  \$/kWh [48]<sup>1</sup>,  $F_c$  [ℓ] is the ICE fuel consumption with associated cost  $\beta_f = 1.08$  \$/ℓ<sup>2</sup>,  $\text{SoH}_b^{\text{end}}$  is the final LIB SoH,  $C_b$  [kWh] is the LIB capacity,  $\beta_{\text{LIB}} = 124.24$  \$/kWh [38] is the LIB pack price and  $L_d = \int_0^{t_{\max}} v_x$  is the distance.

## A. RESULTS FOR THE BRAZILIAN REAL WORLD DRIVE CYCLE

By applying the CEEMD algorithm to decompose the power demand in the Brazilian real-world drive cycle,  $K = 28$  IMFs were obtained. Some of the produced IMFs are shown in Fig. 12a, while the CEEMD algorithm diagram is shown in Fig. 12b. Higher orders IMFs are sequences of lower frequency, which can be confirmed by analyzing the PE in Fig. 13. The  $\text{IMF}_k$  for  $k \geq 22$  present PE less than the white noise standard deviation  $\sigma$  and, therefore, are used to calculate the low frequency demand.

<sup>1</sup>1€ for 1.11\$

<sup>2</sup>Brazilian average price in September 2nd, 2024

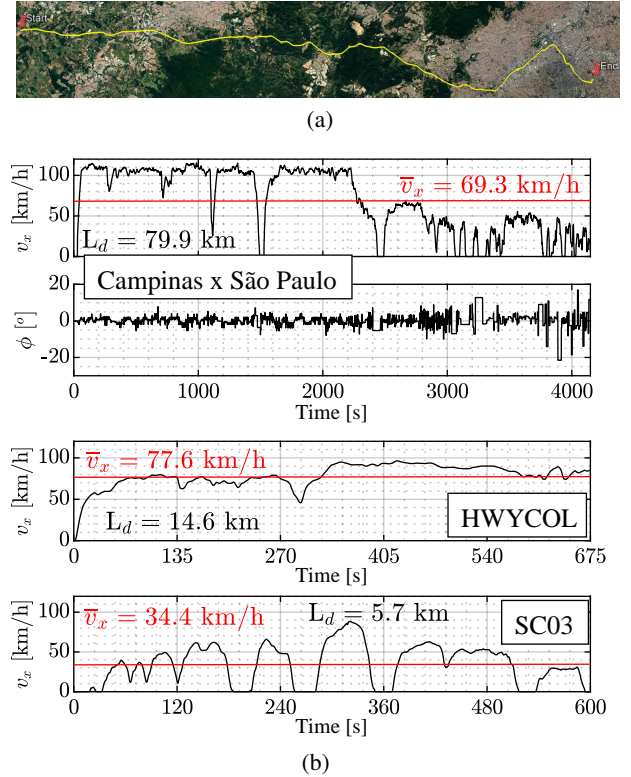


FIGURE 11. Drive cycles: (a) Brazilian drive-cycle path [39] and (b) speed profile for all drive cycles.

The first scenario compares the nominal operation with ICE+LIB, while the second scenario considers the LIB is the main source, with  $P_{\text{ICE}}^*$  set as the minimum value  $P_o$  in (33). As can be seen in Fig. 14, the UC power presents faster dynamic compared to the LIB power. When the LIB is the main source, it delivers a higher power, which leads to an excessive discharge of the device, reaching  $\text{SoC}_b \leq \text{SoC}_{\min}$ . Consequently, the LIB temperature increase and the SoH is strongly reduced in relation to the nominal operation (LIB lifetime 23 times smaller). Moreover, the UC and dc-link voltages are restored to their reference values. Additionally, the LIB voltage and current, the UC current and the butterfly valve angle are shown in Fig. 15. The LIB  $c$ -rate is 1.28 times bigger when the ICE is underutilized, which increases the LIB usage and stress.

## B. UC VOLTAGE RESTORATION

To verify the effectiveness of the proposed strategy under other drive cycles and evaluate the UC terminal voltage restoration, the PLC was compared with traditional LPF and PI methods found in the literature under the HWYCOL drive cycle in numerical results performed in Matlab® and Simulink® environment. Although the UC voltage integral absolute error (IAE) is lower for the LPF and PI strategies (see Fig. 16b), the LIB power presents higher dynamic (see Fig. 16a). In addition, the LIB stress is reduced with the PLC by relaxing the voltage restoration, demonstrated by



TABLE 2. HEV main parameters.

Parameter	Symbol	Value
Vehicle		
Total mass	$M_v$	4,000 kg
Aerodynamic drag	$C_d$	2
Front area	$A_v$	7.0 m <sup>2</sup>
Density	$\rho_a; \rho_f$	1.29; 815 kg/m <sup>3</sup>
Shaft stiffness	$K_{sh}; K_{sl}$	131,780; 487,587 Nm
Shaft damping	$B_{sh}; B_{sl}$	4,666; 17,266 Nms
Reducer efficiency	$\eta_r$	95%
Reducer inertia	$J_r$	0.005 kgm <sup>2</sup>
Load inertia	$J_m$	80.125 kgm <sup>2</sup>
Load friction	$B_m$	0.005 Nms
Wheels radius	$r_d$	0.28 m
Tire ratio rate	$r_L/r_d$	0.98
Slip stiffness	$c_k$	16
Friction coefficient	$f_F$	0.020
Reducer ratio	$i_r$	3.7
Lithium-ion battery and Ultracapacitor		
LIB capacity	$C_b$	120 Ah
Polarization capacitor	$C_1$	15,721.5 F
Heat conduction	$R_c; R_u$	182.36; 1,410 KW <sup>-1</sup>
Thermal capacitances	$C_c; C_s$	10 kJK <sup>-1</sup> ; 50 kJK <sup>-1</sup>
UC capacitance	$C_{uc}$	2 F
UC resistance	$R_{ou}$	50 mΩ
Internal combustion engine and generator		
Maximum power	$P_e^{\max}$	50 kW
Velocity for $P_e^{\max}$	$\omega_{pp}$	5,000 rpm
Model coefficients	$\gamma_1$	0.6526
	$\gamma_2$	1.6948
	$\gamma_3$	-1.3474
Valve dc source	$v_s$	12.0 V
dc motor gain	$K_m$	19.3 rpm/V
Chopper PWM gain	$K_c$	0.12 V/%
Motor time constant	$\tau_m$	0.052 s
Valve delay	$\tau_v$	0.01 s
Rectifier efficiency	$\eta_r$	97%
Inertias	$J_e; J_g$	0.005 kgm <sup>2</sup>
BBC converters and dc-link		
Inductances	$L_b; L_u$	10 mH; 5 mH
Inductor resistances	$r_b; r_u$	100 mΩ; 50 mΩ
Switches resistance	$r_s$	30 mΩ
Switching frequency	$f_s$	20 kHz
dc-link capacitance	$C_{dc}$	3,000 μF

TABLE 3. EMS and control parameters.

Parameter	Symbol	Value
ICE PI <sub>p</sub> controller	$K_p^p; K_i^p$	1 $\frac{\text{rpm}}{\text{W}}; 3 \frac{\text{rpm}}{\text{Ws}}$
ICE PI <sub>ω</sub> controller	$K_p^ω; K_i^ω$	0.5 $\frac{\text{rad}}{\text{rpm}}; 2 \frac{\text{rad}}{\text{srpm}}$
ICE PI <sub>θ</sub> controller	$K_p^θ; K_i^θ$	0.1 $\frac{1}{\text{rad}}; 0.3 \frac{1}{\text{srad}}$
LIB PI <sub>b</sub> controller	$K_p^{ib}; K_i^{ib}$	0.1 $\frac{1}{\text{A}}; 1.0 \frac{1}{\text{sA}}$
dc-link PI <sub>v<sub>dc</sub></sub> controller	$K_p^{v_{dc}}; K_i^{v_{dc}}$	1.0 $\frac{1}{\Omega}; 3.0 \frac{1}{\Omega\text{s}}$
UC PL <sub>v<sub>uc</sub></sub> controller	$\beta_u; \tau_u$	10; 10 rad/s
UC PI <sub>i<sub>uc</sub></sub> controller	$K_p^{i_{uc}}; K_i^{i_{uc}}$	0.1 $\frac{1}{\text{A}}; 1.0 \frac{1}{\text{sA}}$
LIB SoC limits	SoC <sub>min</sub> ; SoC <sub>max</sub>	0.4; 0.8
Sigmoid steepness	$\kappa; \kappa_m$	20; 0.05
PSO coefficients	$c_1; c_2; N_p; \Upsilon_m; \Upsilon_s$	2.05; 2.05; 40; 2; 5

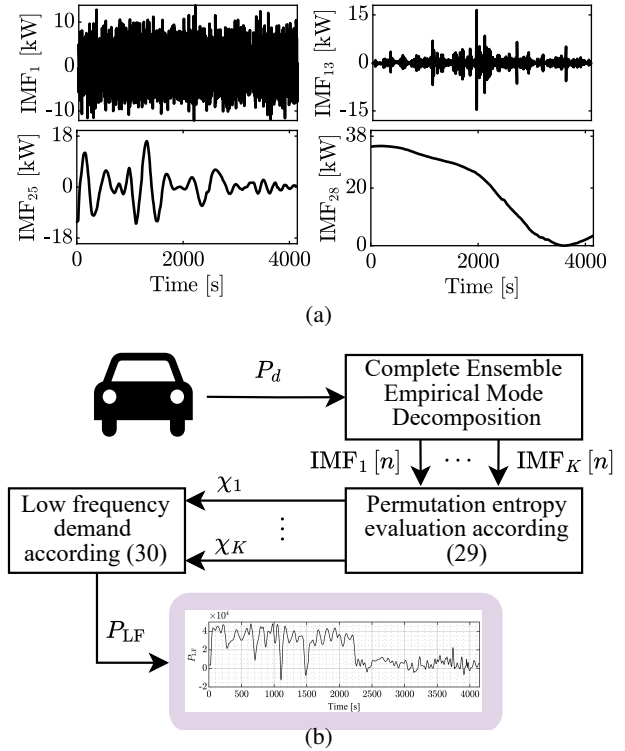


FIGURE 12. CEEMD algorithm: (a) IMF<sub>1</sub>, IMF<sub>13</sub>, IMF<sub>25</sub> and IMF<sub>28</sub> and (b) low frequency demand evaluation.

the lowest standard deviation  $\sigma_w$  of the LIB power detail coefficient  $D_w$  by using the WT (see Fig. 16e, Fig. 16f and Fig. 16g).

The LIB voltage, current, and duty cycle are presented in Fig. 17, alongside the corresponding UC current and duty cycle. It is evident that the dynamic behavior of the UC current is more pronounced than that of the LIB, as expected. Furthermore, the LIB current peak observed for the LPF and PI strategies is nearly twice that of the PLC strategy (zoomed-in region in Fig. 17b), which explains the reduced stress associated with the PLC strategy.

**Remark 4:** The PI controller for the UC voltage restoration comparison was tuned for a 20 dB/decade attenuation up to 0.4 Hz to balance low-frequency gain and cut-off frequency,

with  $k_p = 2.0$  and  $k_i = 5.0$ . Similarly, the LPF time constant was set to a frequency 0.05 Hz.

### C. COMPARISON WITH THE LITERATURE

In this section, the proposed SPSO-CEEMD strategy is compared with other methods in the literature. The IT2-FLC in [15], originally designed for a P-HEV, was adapted to application in the S-HEV. As shown in Fig. 18a, the dc-link control and UC management are the same as the proposed in Section III-D. A traditional FLC estimates the optimum ICE torque ( $\tau_{ICE}^{op}$ ) according to the power demand  $P_d$  and the IT2-FLC defines the gain  $\alpha \in [0 \ 1]$  that split the torque demand between the LIB and the ICE. Moreover, the XOS in [16] was also adapted, as shown in Fig. 18b. For the SC03 drive cycle, the IT2-FLC strategy charge more the LIB with the ICE during the drive cycle, which leads to the more

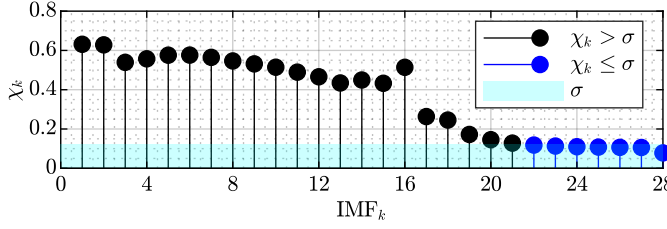


FIGURE 13. PE of each IMF obtained by the CEEMD algorithm.

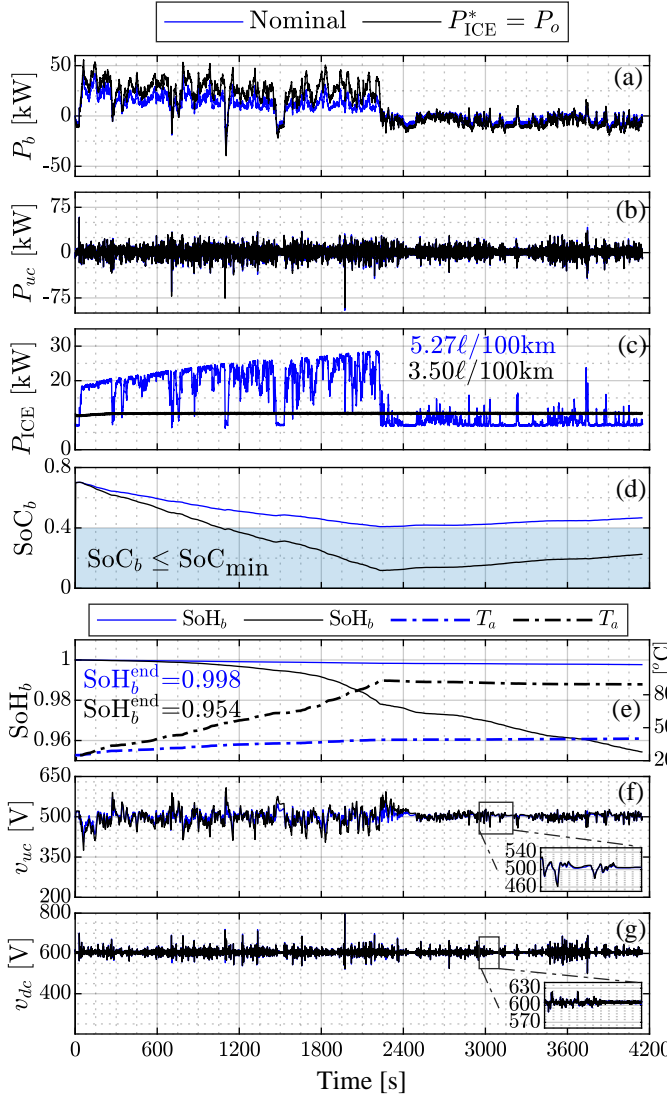


FIGURE 14. Brazilian real-world drive cycle experimental results: (a) LIB power, (b) UC power, (c) ICE power, (d) LIB SoC, (e) LIB SoH and temperature, (f) UC voltage and (g) dc-link voltage.

fuel consumption (see Fig 19a and Fig 19c). However, the ICE restarts are reduced in relation to the XOS strategy (see Fig 19d). The proposed SPSO-CEEMD strategy keeps the ICE on during all the with intermediary fuel consumption.

Figure 20a confirms that the LIB is discharged often with the IT2-FLC strategy, which leads to a positive LIB energy

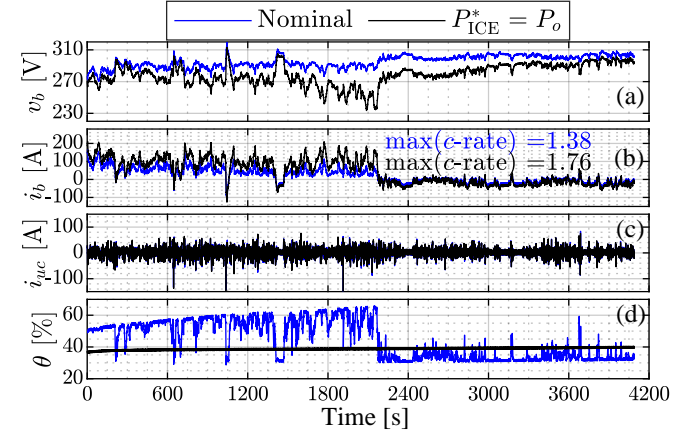


FIGURE 15. Brazilian real-world drive cycle experimental results: (a) LIB voltage, (b) LIB current, (c) UC current and (d) butterfly valve angle.

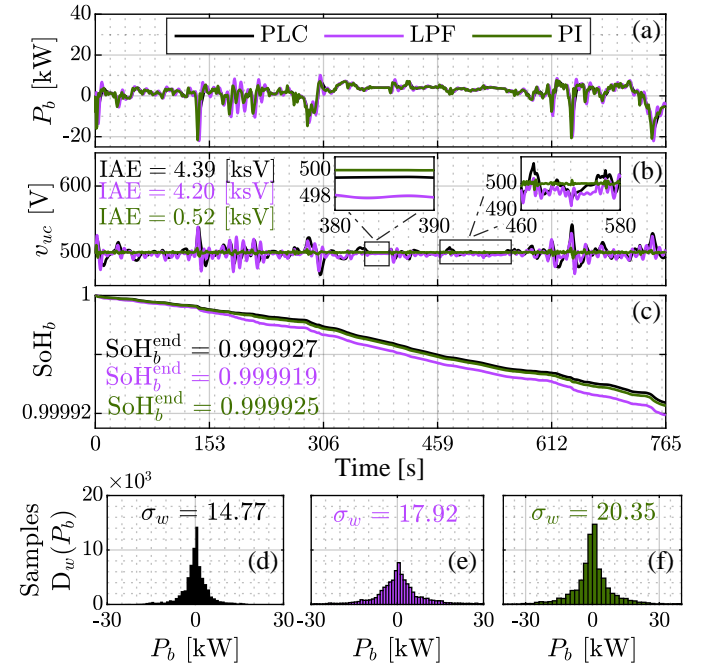
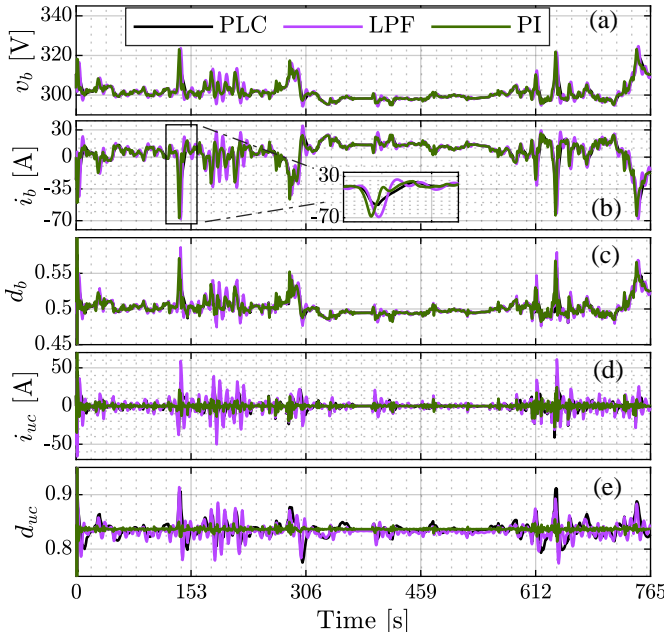
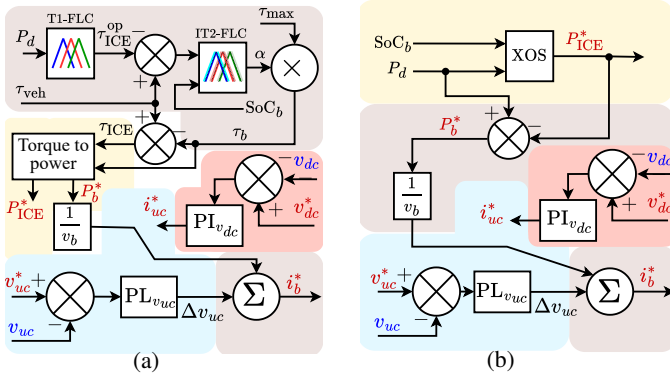


FIGURE 16. UC voltage restoration comparison under the HWYCOL drive cycle: (a) LIB power, (b) UC voltage, (c) LIB SoH and Wavelet standard deviation of the detail coefficient using (d) PLC, (e) LPF and (f) PI

consumption (see Fig. 20k). The value  $SoH_b^{end}$  is bigger and the final temperature  $T_a^{end}$  is lower with the proposed SPSO-CEEMD (see Fig. 20b, Fig. 20c), indicating a larger lifetime. Analyzing the LIB power detail coefficients  $D_w$  obtained by the WT (see Fig. 20d, Fig. 20e, Fig. 20f) and the standard deviation  $\sigma_w$  (see Fig. 20g, Fig. 20h, Fig. 20i), it can be seen that the LIB stress is smaller with the proposed strategy. Moreover, the total cost in (42) for the proposed strategy is the lowest (see Fig 20l).



**FIGURE 17.** UC voltage restoration comparison under the HWYCOL drive cycle: (a) LIB voltage, (b) LIB current, (c) LIB duty-cycle, (d) UC current and (e) UC duty-cycle.

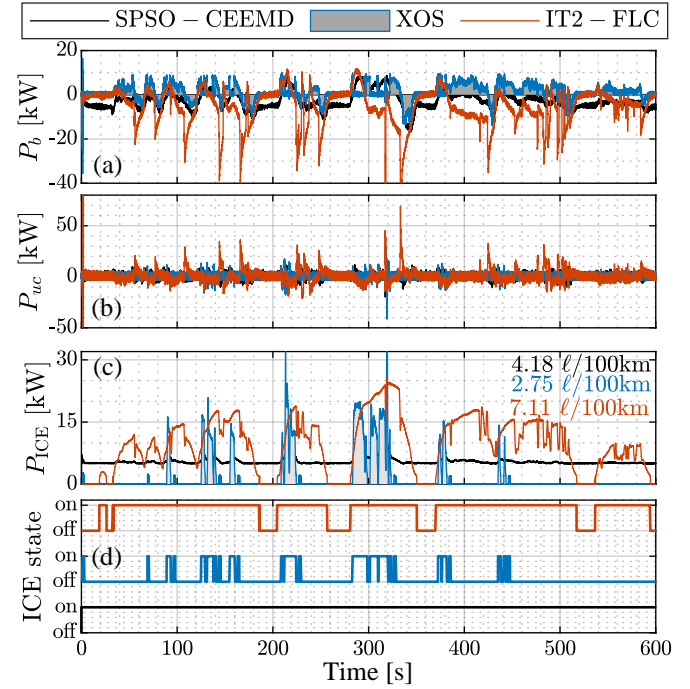


**FIGURE 18.** Block diagrams: (a) IT2-FLC adapted from [15] and (b) XOS adapted from [16].

#### D. TRADE-OFF BETWEEN FUEL SAVING AND EMISSIONS

In this section, the CO and NO<sub>x</sub> emissions are taken into account by adding the constraints in the optimization problem in (41) for three different values of  $k_{CO}$  and  $k_{NO_x}$ . The ICE and LIB powers are shown in Fig. 21, where it is possible to see that for greater values of  $k_{CO}$  and  $k_{NO_x}$  the emission constraint is more relaxed, making the ICE be more used than the LIB. However, when the emission constraint is hard  $k_{CO} = k_{NO_x} = 0.5$ , the LIB power is greater.

To assess the effectiveness of emission optimization, Fig. 22 presents the average emissions of the constrained SPSO-CEEMD for different values of  $k_{CO}$  and  $k_{NO_x}$ , alongside the unconstrained SPSO-CEEMD and the IT2 and XOS



**FIGURE 19.** Experimental comparison for the SC03 drive cycle: (a) LIB power, (b) UC power, (c) ICE power and (d) ICE state.

methods adapted from the literature. As can be seen, the constraints on CO and NO<sub>x</sub> emissions are satisfied for all values of  $k_{CO}$  and  $k_{NO_x}$ . Moreover, the numerical results for all scenarios are presented in Table 4, which demonstrates that emission reduction leads to increased LIB usage. For instance, setting  $k_{CO} = k_{NO_x} = 0.5$  results in a 44.21% reduction in CO emissions and a 44.78% reduction in NO<sub>x</sub> emissions. However, the final SoH analysis indicates that the LIB lifetime is 1.44 times longer when emission constraints are not considered.

#### V. CONCLUSION AND FUTURE WORKS

In this work, a CEEMD strategy is applied for the first time in a HEV to improve the LIB lifetime by avoiding high frequency demand during driving. The ICE is managed by sigmoid functions optimized by the PSO algorithm considering the trade-off between fuel consumption and LIB SoH, while incorporating the emission reduction as constraints of the optimization.

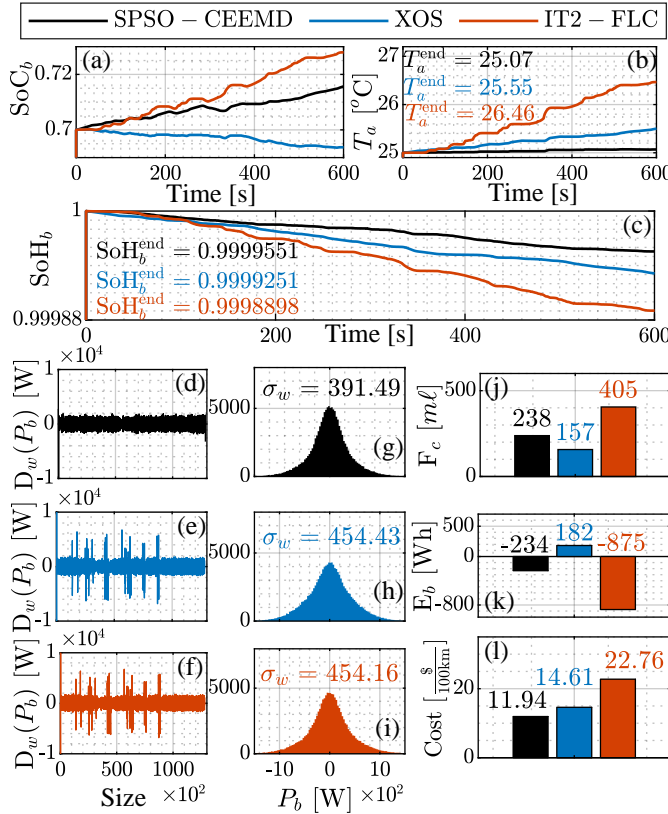
The ICE control is achieved through a triple control loop strategy, which regulates power, angular velocity, and the angle of the butterfly valve. The UC is responsible for the dc-link voltage control, with its terminal voltage restored by a PLC to keep the device available in the next power peaks demand.

The theoretical assessments, numerical and experimental results show that:

- 1) The CEEMD algorithm decomposes the power demand without definition of a basis function. With the proposed strategy, it is possible to use the CEEMD in

**TABLE 4.** Energy management comparison: numerical results.

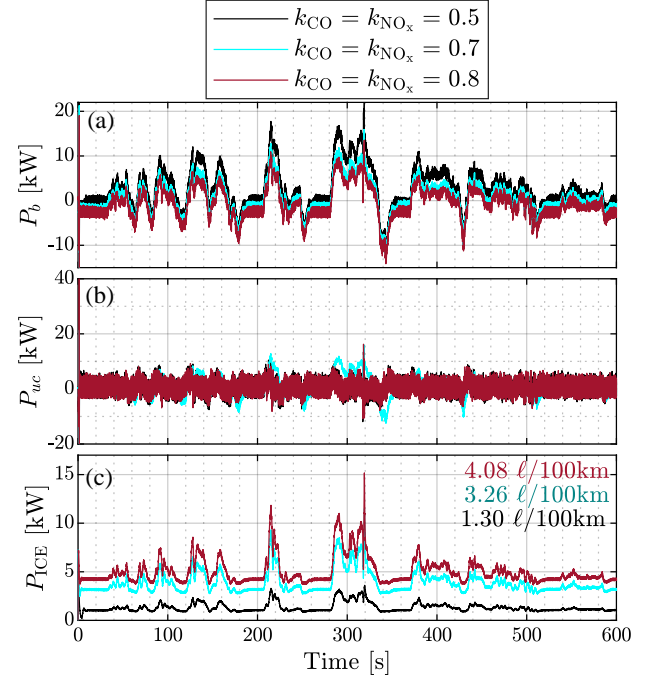
		Fuel [ $\ell/100\text{km}$ ]	SoH <sub>b</sub> <sup>end</sup>	CO [g/km]	NO <sub>x</sub> [g/km]	max( $T_a$ ) [°C]	max( $c$ -rate)	F <sub>c</sub> [m $\ell$ ]	Converters' efficiency [%]
Constrained SPSO-CEEMD	$k_{CO} = k_{NO_x} = 0.5$	1.30	0.9999354	1.83	0.37	25.20	0.69	73.96	97.12
	$k_{CO} = k_{NO_x} = 0.7$	3.26	0.9999495	2.52	0.57	25.17	0.61	186.10	97.31
	$k_{CO} = k_{NO_x} = 0.8$	4.08	0.9999500	2.90	0.63	25.13	0.54	232.30	97.37
Unconstrained SPSO-CEEMD		4.18	0.9999551	3.28	0.67	25.07	0.45	238	97.42
IT2-FLC		7.11	0.9998898	4.22	0.86	26.46	1.42	405	97.06
XOS		2.75	0.9999251	2.43	0.49	25.55	0.93	157	97.09



**FIGURE 20.** Experimental comparison for the SC03 drive cycle: (a) LIB SoC, (b) LIB temperature, (c) LIB SoH, Wavelet detail coefficient using (d) SPSO-CEEMD, (e) XOS, (f) IT2-FLC, detail coefficient standard deviation using (g) SPSO-CEEMD, (h) XOS, (i) IT2-FLC, (j) fuel consumption, (k) LIB energy and (l) cost.

a HEV to define the low frequency component as reference for the LIB and ICE, which reduces the LIB aging and allow the ICE control;

- 2) The sigmoid functions allow the ICE control considering the butterfly valve slow dynamic and avoid discontinuities. Furthermore, the sigmoid optimization by the PSO algorithm reduces the fuel consumption and LIB aging for a real world Brazilian drive cycle, the SC03 and the HWYCOL drive cycle, which showed the strategy adaptability;
- 3) The numerical results for the HWYCOL drive cycle showed that the phase-lag compensator can restore the



**FIGURE 21.** Experimental results for the SC03 drive cycle considering the emission constraints for different values of  $k_{CO} = k_{NO_x}$ : (a) LIB power, (b) UC power and (c) ICE power.

UC terminal voltage without affect its power deliver in high power demands when compared to the PI and LPF strategies by relaxing the voltage restoration.

- 4) Although the IAE is the largest with the PLC, the LIB power dynamic is not highly affected by the UC voltage restoration. This is verified by the lowest wavelet detail coefficient standard deviation and the highest final SoH, indicating a lifetime 2.74% and 10.96% increased compared to traditional PI and LPF methods, respectively;
- 5) The experimental results demonstrated a 18.28% and 47.54% lower total operational cost compared to XOS and IT2-FLC strategies adapted from the literature. Moreover, for the SC03 drive cycle, the Haar Wavelet decomposition indicates a lower LIB stress with the proposed strategy and a LIB lifetime about 1.67 and 2.45 times higher than the XOS and IT2-FLC strategies, respectively, according the final SoH;



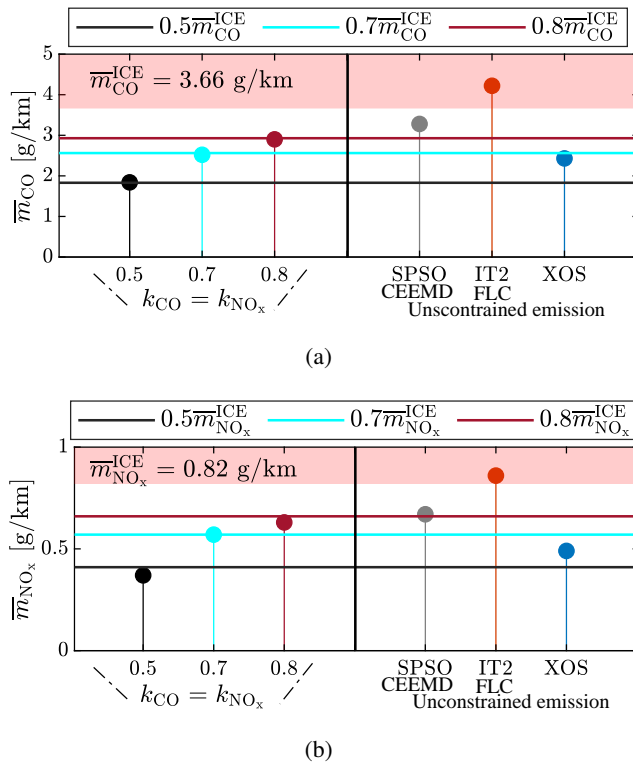


FIGURE 22. Average emissions under the unconstrained optimization and the constrained optimization for different values of  $k_{CO}$  and  $k_{NO_x}$ : (a) CO emission and (b) NO<sub>x</sub> emission.

6) Considering the emission constraints in the optimization problem, the experimental results indicate that emissions can be reduced depending on the optimization parameter selection. However, emission reduction leads to increased LIB usage, as reflected in the final SoH analysis, where the LIB lifetime is extended by a factor of 1.44 when emission constraints are not considered.

The simplicity and efficacy of the proposed SPSO-CEEMD strategy position it as a promising benchmark for EMSs. Future work includes incorporating the ICE gas emissions pricing according the carbon credit in the optimization and enhancing the proposed strategy by integrating driving speed prediction, enabling real-time optimization.

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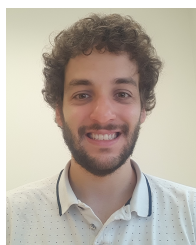


converters for renewable energy sources and storage systems.



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